INSTRUCTION BOOK

No. 27-12

FOR

INSTALLING AND OPERATING

ATLAS IMPERIAL

FOUR CYCLE

MECHANICAL INJECTION

DIESEL ENGINES

ATLAS IMPERIAL DIESEL ENGINE CO.

OAKLAND, CALIFORNIA, U. S. A.

CABLE ADDRESS "ATGAS"

CODES

BENTLEY'S

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Explanation of the Diesel Principle

FACTS:

1. Air becomes hot by being compressed. The degree of heat depends on the amount of compression.

2. Three hundred and seventy-five pounds compression creates sufficient heat to ignite fuel oil when properly sprayed into the com-

pression chamber at the proper time.

3. Fuel oil, if sufficiently atomized, will ignite when injected into a cylinder of compressed air, the temperature of which has been raised above the fire test of the fuel by means of compression.

I. The expansion of the burning fuel oil is the power-creating force

within the cylinder of the engine.

The diesel engine differs from a gas engine from the fact that it is a constant pressure engine. That is, the injection of the fuel into the cylinder is timed and controlled to maintain constant pressure during its introduction, while in a gas engine a constant volume of mixture is taken into the cylinder and after compressing same to a safe limit (about 50 pounds) the mixture is ignited by some auxiliary mechanism and the pressure is instantly raised in the nature of an explosion to a degree depending on the volume of the mixture.

While the terminal pressures are about the same in both types of engines, the efficiency of the diesel is twice that of the gas engine, due to compressing the air to a high temperature before injecting the fuel, then controlling this injection so that combustion continues for a pre-

determined time, varying with the load.

This process enables the diesel engine to maintain a much higher mean effective pressure with a corresponding greater horsepower for the same cylinder dimensions.

In order to obtain perfect combustion within the cylinder it is necessary to inject the fuel oil in the form of a very fine spray which

is done by the specially designed spray nozzle.

By understanding these fundamental requirements of a diesel engine it is readily seen that the mechanical injection diesel engine is very simple.

CYCLE:

Atlas-Imperial Diesel Engines operate on the four cycle principle. The four cycle implies that the work in the cylinder is accomplished in four strokes of the piston, or two revolutions of the engine.

The function of each cycle or stroke of the piston is as follows:

1. INTAKE STROKE:

When the piston goes down on the intake stroke, the intake valve being open, the cylinder becomes filled with air from the atmosphere.

2. COMPRESSION STROKE:

On the upward stroke of the piston, the inlet valve is closed and the air in the cylinder is compressed to about 375 pounds pressure per square inch. At this pressure (as explained above) the temperature of the compressed air is raised sufficiently to ignite the fuel. A few degrees before the piston reaches the top of the compression stroke the fuel spray valve is opened by a cam and the fuel sprayed into the heated

air. A constant pressure of fuel oil is always maintained in the spray nozzles at all times by means of small plunger pumps which are more fully described later in the book.

3. EXPANSION STROKE:

The fuel oil burning and expanding in the cylinder maintains pressure on the piston during the downward stroke. Near the end of the power stroke the exhaust valve opens and allows the burned gases to escape.

4. SCAVENGING STROKE:

As the piston returns to the top of the cylinder with the exhaust valve open, the piston pushes all of the burned gases out of the cylinder through the exhaust valve. When the piston has reached the top, the exhaust valve is closed and the intake valve again opened.

This completes the four cycles or four strokes of the ATLAS-IMPERIAL MECHANICAL INJECTION DIESEL Engine and describes the method of its functioning.

It will be noted that while the sequence of the strokes forming this cycle is similar to that of the gas engine, the functions performed during the cycle (except on the exhaust stroke) are materially different.

On the first stroke, by taking into the cylinder pure air only, we are enabled to compress this charge sufficiently to secure a very high temperature which would be impossible with a charge of gas mixture, because of liability of preignition.

This high temperature secured allows the use of cheap, low grade fuel oils of high fire test which are safe to handle and increases both the thermal and commercial efficiency of the engine.

On the third stroke the fuel is introduced under absolute control, as to timing and quantity, the constant pressure in the column of fuel oil being sufficient so that when the valve opens the fuel is completely atomized and ready for instant firing at the moment it reaches the heated interior of the cylinder. No high pressure air is used to force this fuel oil in the cylinder.

With some methods (not true solid injection type) fuel is introduced under low pressure into a recessed casting, projecting into the firing chamber while there is no pressure, and on the up stroke of the piston the pressure increases until sufficient heat is developed to ignite the little cartridge of fuel (previously injected under no pressure) and when it ignites it blows the fuel oil out through small holes provided for that purpose. No control of the exact moment of firing is possible, as in our method where the fuel is mechanically injected into the cylinder at the proper time.

As stated above, this combustion is not in any sense an explosion, but takes place during a well defined, predetermined portion of the power stroke and at constant pressure, and because of the nature of its introduction into the cylinder and the large volume of pure air into which it is forced, the combustion is perfect within the range of cylinder power rating, hence the high thermal efficiency of the diesel engine on low grade fuel oils.

The foregoing description of the engine is, of course, general in character, intended to cover only the principles around which the physical construction is assembled.

In General

In issuing this instruction book we have tried to present to the reader the principal points to be observed in the care, operation and maintenance of the Atlas-Imperial Diesel Engine in a clear and concise manner.

This information is intended not only for use by the operator but also for those responsible for the performance of the engine. It is impractical to cover the exceptional conditions which might occur and therefore we have only covered those with which each operator should be thoroughly familiar.

Along these lines we offer as a satisfactory suggestion:

WHEN THE ENGINE IS RUNNING SATISFACTORILY AND SMOOTHLY DO NOT CONTINUALLY TRY TO BETTER THE OPERATION WITH MINOR ADJUSTMENTS.

FIRST: NEVER ALLOW YOUR ENGINE TO SMOKE:

A diesel engine, oil engine, gas engine, kerosene lamp, or a candle perform the same operation, namely: Each burns a certain amount of fuel when mixed with proper proportions of air, and when the proportions are such that there is not enough air for the fuel to burn clean—in other words, too much fuel for the amount of available air—you will have smoke. Whether it be a candle, kerosene lamp, gas engine, oil engine, or diesel engine, smoke indicates that the proportions of air and fuel are not correct. These proportions must be adjusted to a point at which smoke is eliminated,—combustion perfect and exhaust clear.

In a diesel engine there are two things that will cause the engine to smoke; first, as above stated, too much fuel for the amount of air; and secondly, not the right distribution of broken up fuel in the combustion chamber. If the spray nozzle should be out of order or partly clogged, the fuel may enter the combustion chamber in a solid liquid form, in which case it does not properly mix with the air in the cylinders. When the exhaust from an engine is smoky it clearly indicates that combustion is not perfect and that the residue, in shape of smoke, is clinging to the oily surfaces of the cylinders, pistons, piston rings, valves, etc., and when this happens you are creating trouble for yourself and doing an injustice to the engine. Therefore, the first thing in consideration of the operation of a diesel engine is, NOT TO ALLOW YOUR ENGINE TO SMOKE.

SECOND: DO NOT OVERLOAD YOUR ENGINE:

A diesel engine is unlike a gas engine in that it can be overloaded. You can continue increasing the power of a diesel engine to a great extent by admitting too much fuel which increases the heat, therefore increases the pressure in the cylinder. One cannot continually overload an engine any more than you can continually overload any other thing, thinking that you can go on successfully carrying the overload without bad results. Atlas-Imperial Diesel Engines are of such liberal dimensions, that they will pull a considerable overload without smoking; but if one attempts to overload his engine too much it will naturally smoke

continually up to its full power as long as its exhaust is clear and the combustion perfect. Do not allow spray valves, air starting valves, inlet or exhaust valves to be leaky. If an engine has been allowed to smoke for some time, carbon will be deposited on the valve seats in spots and allow the valves to leak. As soon as a valve is leaky a blast of fire will pass through the leaky valve and start to cut the valve seats or valves, thereby causing trouble.

THIRD: LUBRICATION:

Any piece of machinery needs proper lubrication and a diesel engine is not an exception. Always see that lubricating systems are always in proper working order. We have attempted to make the lubricating systems in our engines as near fool proof as possible, but it still remains for human intelligence to watch it from time to time to see that it functions in the way that it should.

The lubricating oil consumption is a variable factor, depending somewhat on the grade of oil used, but principally on the care given the engine in its operation, ordinarily about 750 to 1000 horse power

hours per gallon.

CYLINDERS:

Oil is supplied to the cylinder walls by means of the force feed lubricator through a small pipe connected to the inlet and exhaust side of the cylinder wall. It requires not more than 13 or 14 drops of oil to the cylinder wall per minute. It is very important that this lubrication be constant and does not vary in any way. This may be detected very readily by counting the drops as they drop past the sight glasses located on top of the oiler and should be checked at regular intervals.

BEARINGS:

The lubrication of the main bearings is accomplished by means of a force feed system, so arranged that the oil is forced, by means of a pump at a pressure varying from five to ten pounds, according to that which has been regulated by the pressure lubricating oil relief valve. The oil is pumped from the oil reservoir or supply tank then through oil pipes in the base (on the exhaust side of the engine) to the bottom of the main bearings. This is not done through the top of the bearing caps, thus permitting a simple arrangement of the oil piping and obviating the necessity of breaking pipe connections, etc. The crank shaft revolving in the main bearing is provided with a hole drilled angular through to the crank pin, through which the oil is forced, thereby lubricating the connecting rod bearings or crank pins.

It is important that sufficient oil be always maintained in the oil reservoir to insure the proper function of the pumps so that the oil will circulate freely, unhampered by air bubbies. A pressure gauge is connected to the lubricating oil pipe line, which indicates the oil pressure on the bearings at all times. If no pressure is registered on this gauge, it will indicate one of three things: ONE—that there is not sufficient oil in the oil reservoir; TWO—that the lubricating oil pumps are

not functioning properly; or THIRD—that some bearing has become sufficiently loose to allow the lubricating oil to flow out too freely.

All the oil pumped through these bearings is collected in the crank pits, flows to a sump through a strainer, is pumped from the sump through a filtering tank, then back to the main bearings.

WRIST PINS:

In the top half of the crank bearing at the center of the groove is a hole for the oil to leave the crankbearing and enter the connecting rod. The connecting rod is drilled hollow and is fitted with a check valve at its lower end, (which checks any attempt of the oil to return back into the crank) so that the oil which is forced through the crankshaft and through the hole in the bearing will pass through the check valve and then up through the hollow connecting rod to lubricate the wrist pin in the piston. (See section drawing on next page.)

OIL RETURN:

During its course the oil is being gradually squeezed out, partly through the main bearings, partly through the crank bearings and the balance out through the wrist pin bearings, from where the oil returns by gravity to the sump in the bottom of the base.

The lubricating oil distributing pipe is provided with a springloaded relief valve, so arranged as to allow the surplus oil to be delivered back into the lubricating oil reservoir and thereby preventing the pressure from becoming excessive.

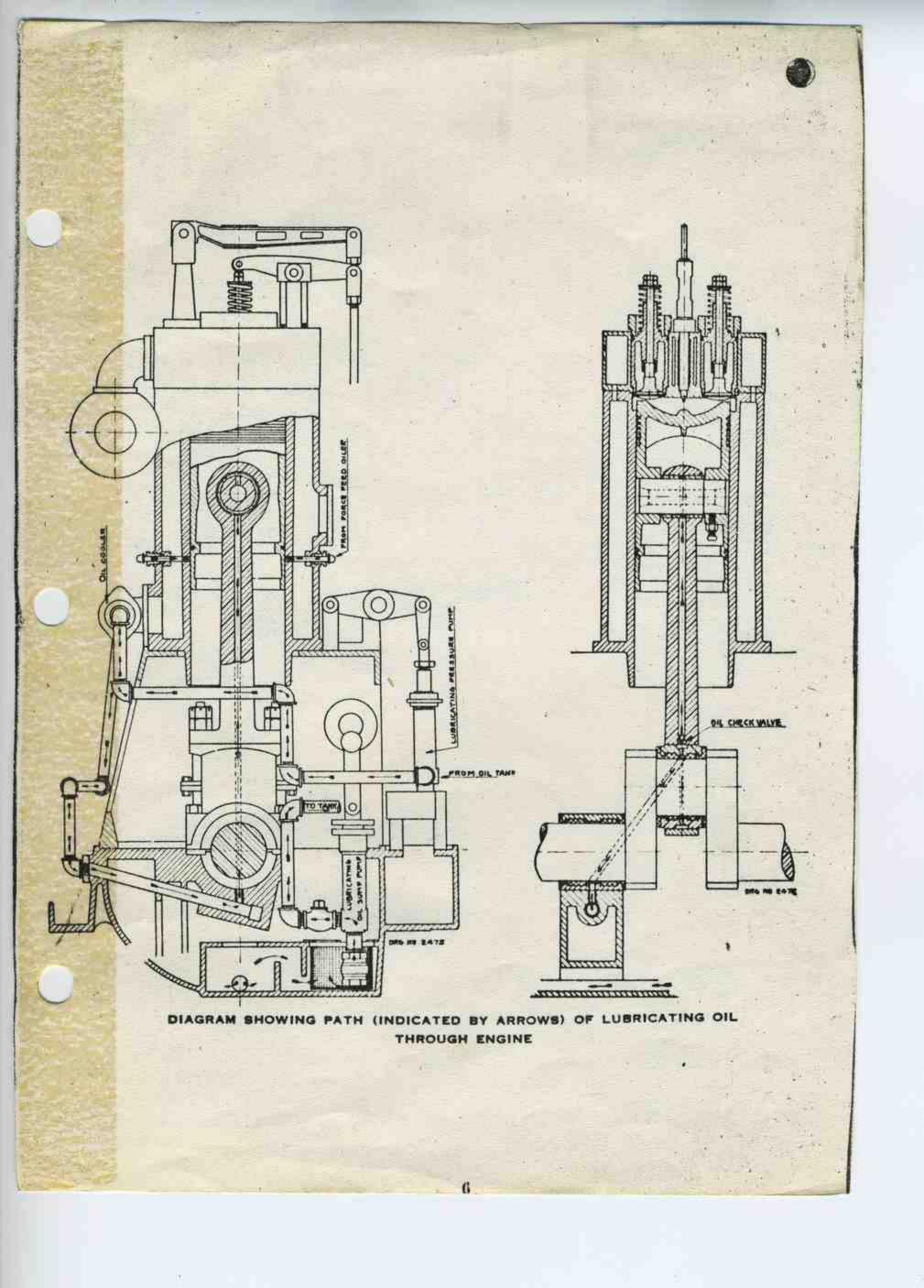
CHOICE OF OIL:

When the engine warms up after running for some time, the oil pressure may not register as high as when the engine was first started, due to the fact that the oil becomes thin with heat, and therefore flows more freely out through the sides of the bearings.

The choice of lubricating oil for the power cylinders and bearings of a diesel engine is a matter involving many operating factors of which the most important are the temperature and pressures within the power cylinders, the thoroughness of oil distribution over the working surfaces, the amount of carbon formation and the method of lubrication.

Fairly high temperatures and pressures are to be contended with in a diesel engine. The oil film formed, therefore, must have sufficient body to support the piston and piston rings, preventing metallic contact with the cylinder. Heat will naturally thin out the oil and therefore in the selection of the lubricant the body of the oil at the operating temperatures must be considered. High quality, correctly refined diesel engine cylinder oils retain their body better than the low grade, inferior oils.

Unsuitable or inferior oil fails to provide satisfactory lubrication. It chars freely, combines easily with fuel impurities the products of incomplete combustion and dust or dirt in the intake air, forming a hard crust-like deposit. In an attempt to supply lubrication with such an oil excessive feeds to the cylinders are generally employed. This aggravates the difficulties. The oil then works itself in between and behind



the piston rings where it gradually bakes and decomposes forming de-

posits that cause the rings to stick in their grooves.

The trade names of oils suitable for lubricating ATLAS-IMPERIAL DIESEL ENGINES and the names of the companies marketing them are far too numerous to mention in the limits of this book. We have found from our experience that a high grade oil of the following characteristics usually meets all the requirements when selected according to the following guide as to viscosity:

Engine Room Temp. °F S & E Viscosity Number

Below 40
20-60
30-75
50-90
65-100
75-130
85 and over

In the selection of a lubricating oil it must be remembered that only oils of the highest grade should be used. In order to assure yourself on this point, deal with a reputable oil company. Most oil companies maintain lubricating engineers on their staffs and these men should be consulted freely as they can make recommendations on the ground with the full knowledge of the conditions under which the engine is to work.

Fuel Oils

To assist in obtaining the proper oil to be used as fuel in this engine, we transmit herewith our recommendations. These recommendations are the results of experiments which have been con-

In purchasing the fuel oil to be used for this engine, you should advise the oil company that it will be used in a mechanical injection type diesel engine. We recommend that your first purchase be only a few hundred gallons rather than a large supply, in order to be sure

that you obtain the proper kind of oil.

There are two characteristics of oil which are of EQUAL IMPORT-

ANCE. First, its GRAVITY; and second, its VISCOSITY.

It has been found that low Gravity oils are not satisfactory for all year service as they congeal in cold weather. Similarly it has been found that high Gravity oils, or oils about 38° or higher do not function satisfactorily at all times, especially in the warmer temperatures.

Therefore, the results of the various tests conducted reveal the

following:

1. It has been found that any oil of less than 24° Gravity Baume must be filtered and heated. While this oil may be satisfactorily used in certain localities, generally speaking it is unsatisfactory for the average installation where cold weather is encountered, on account of the difficulty in heating the oil.

 That 24° to 28° Gravity Baume oil is usually clean and needs ne heating or straining before reaching the filter provided on the engine except in extremely cold weather providing the oil has the proper Vis-

cosity.

3. The conclusions reached with regard to engines that must be operated at zero temperatures are that during cold weather periods oils

from 32° to 38° Gravity Baume should be used, but that at all other times the 24° to 28° Gravity Baume oil should be used.

The Viscosity should be from 40 to 50 seconds with a Saybolt test 100° Fahrenheit. If the Viscosity runs to any great degree higher than 50 seconds, you will find that the oil is too thick for use as it is liable to clog the strainers and pumps.

Therefore, specifications for the oil best suited for this engine are as follows:

Gravity-24° to 28° Baume.

Calorific value-18,500 to 19,500 B.T.U.'s.

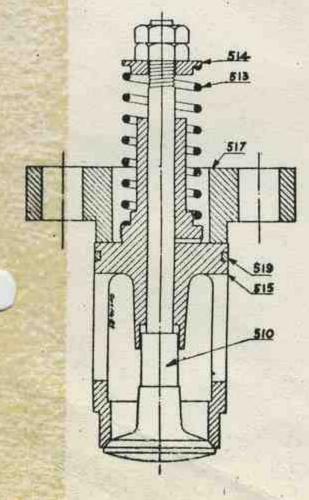
Irrespective of what oil is used, it is necessary that the oil be refined to the extent that all water, sand and grit has been removed.

You will note from the above that the fuel oil most suitable is one from 24 to 28 degrees gravity.

This is the oil which we ordinarily use to test the engine at our plant. It is known as Fuel Oil or Diesel Fuel Oil and it is sometimes incorrectly called Crude Oil.

For temperatures from zero to 15° Fahrenheit above zero it would probably be best to use 32° Baume Gravity Oil, but for sub-zero temperatures you would have to use a 36° Baume to 38° Baume Gravity Oil.

Fuel oils lighter than distillate, such as kerosene, etc., are not suitable for this engine. It is possible that you may be able to adjust the engine so it will function satisfactorily for a short period, but the engine is not intended for such oils and if used trouble probably must be expected later, as such fuels will damage the valves and pumps. This is equally true of any other diesel or semi-diesel engine.



EXHAUST AND INLET VALVE CAGE TYPE

G515	Valve and cage complete. Parts 510, 513,
201	514, 515 and valve nuts.
510	Exhaust or inlet valve.
512	Exhaust or inlet valve cage bushing (only in 11" bore engines and larger)
513	Exhaust or inlet valve spring.
514	Exhaust or inlet valve spring bushing-top
515	Exhaust or inlet valve cage.
517	Exhaust or inlet valve cage flange.
518	Exhaust or inlet valve cage gasket.
519	Exhaust or inlet valve cage ring.

Fuel Oil

Theoretically, a Diesel engine should burn any petroleum product if properly prepared. However, there are certain impurities in almost all fuel oils which determine to a large extent the suitability of those oils for operation in a Diesel engine. The following brief description of the various impurities, with their effect upon the engine, is given to guide the purchaser and the oil supplier in the selection of a proper fuel oil.

GRAVITY

Specific gravity is in itself no indication of the suitability of a Diesel fuel oil.

VISCOSITY

The viscosity of a fuel oil will determine its suitability by the effect that it has on the flow of the fuel oil through the pipes, valves and so forth. An oil of high viscosity that may be in all other respects satisfactory will require heating to enable it to flow satisfactorily through the fuel lines, valves and so forth. Another effect of high viscosity is the resistance that it offers to vaporization, requiring variation in the size of the burner tips.

MECHANICAL IMPURITIES

Under this heading come such elements as dirt, grit, fiber and water. The small holes in the spray valve tips, strainers and so forth are liable to become clogged if there is too large an amount of mechanical impurities. Besides this, they have the effect of cutting out the seats of the valves.

CARBON RESIDUE

This is a characteristic of fuel oil which has only recently received attention, at least in Diesel fuel oil specifications, and one which has quite an effect on the operation of an engine. This carbon residue (Conradson carbon) is a measure of the proportion of carbon deposits that are likely to occur on the piston, cylinder head, valves and so forth. A high percentage of Conradson carbon is usually associated with carbon formation within the cylinders, as well as gumming up of the valves and stems.

LUBRICATING VALUE

All petroleum products have a certain amount of oiliness and the higher this oiliness is in a fuel oil the less wear will be noticed on the pump plungers, spray valve stems and such similar parts. Very light oils, such as kerosene, contain little or no lubricating value and therefore their use will usually result in excessive wear on the pump plungers, spray valve stems and so forth. The following are the general characteristics of a suitable Diesel fuel:

Viscosity (Saybolt Universal) at 100° F 35 to	60 seconds
	maximum
	maximum
THE REPORT AND ADDRESS OF THE PROPERTY OF THE	maximum
	maximum

Legal regulations usually limit the flash to a minimum of 150° F.

Diesel fuels from California crudes are readily obtainable, meeting the above specifications in gravities between 24 and 30° API. Midcontinent fields produce so-called gas oils with gravities ranging from 32 to 36° API. European oils likewise come within this same range.

We cannot place too much emphasis upon the importance of getting oil which is free from impurities to avoid the clogging of valves, strainers, spray valve tips and so forth and upon the desirability of securing an oil with as much oiliness as possible to avoid wear on some of the vital parts of the engine.

DO NOT CONFUSE DIESEL FUEL WITH ORDINARY BOILER OIL. THE LATTER CONTAIN DIRT AND SEDIMENT WHICH MAKE THEM UNSUITABLE FOR A DIESEL ENGINE.

A heavy fuel oil thinned down by the addition of a lighter oil is not a satisfactory fuel, as the lighter oils in such a mixture burn off rapidly and leave the heavier oil to gum up piston rings and valves and cause excessive cylinder wear. Insist on getting a straight run distillate for your fuel.

CARE OF FUEL OIL

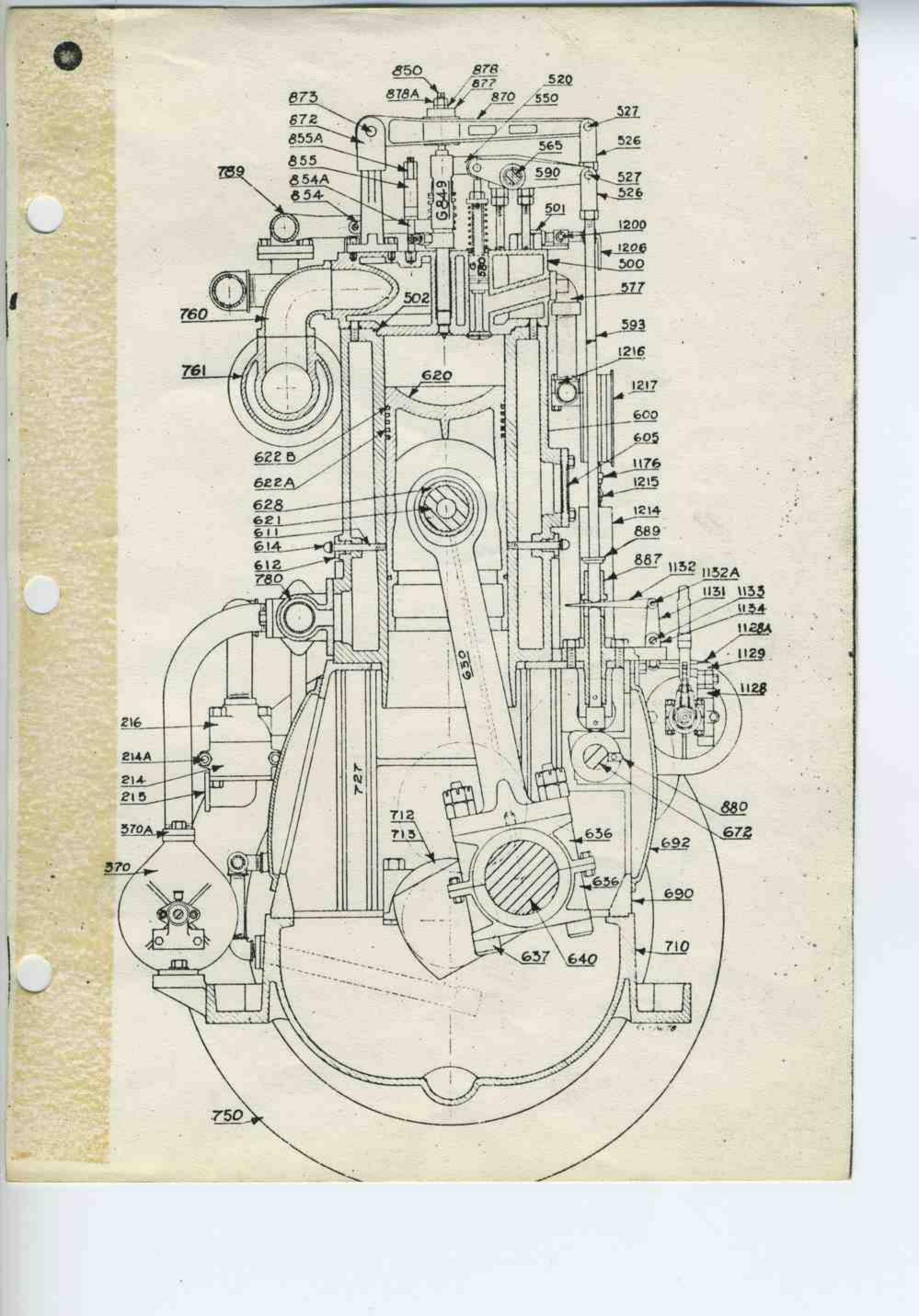
The best fuel oil can be ruined if not properly handled. Store and transport your fuel in clean container and use your containers for fuel oil only. Flush out the container frequently to remove any dirt, rust or scale that may accumulate.

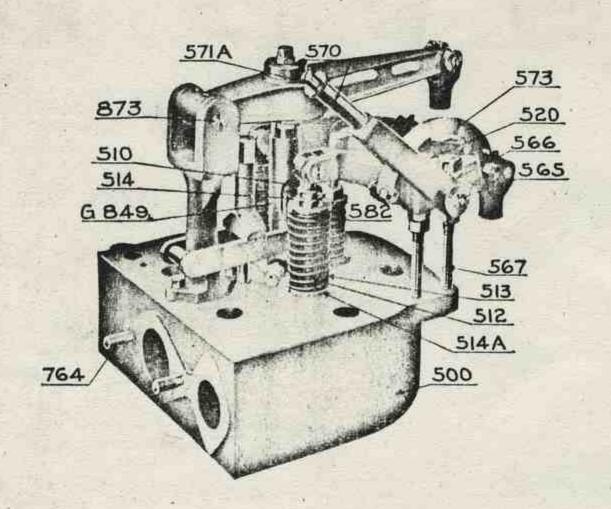
Keep your containers closed to prevent dust, dirt and water getting in the fuel oil, as these impurities in the fuel are bound to cause you more trouble and annoyance than required to keep your fuel clean. Provide means in your storage tanks for draining off any water that may be present.

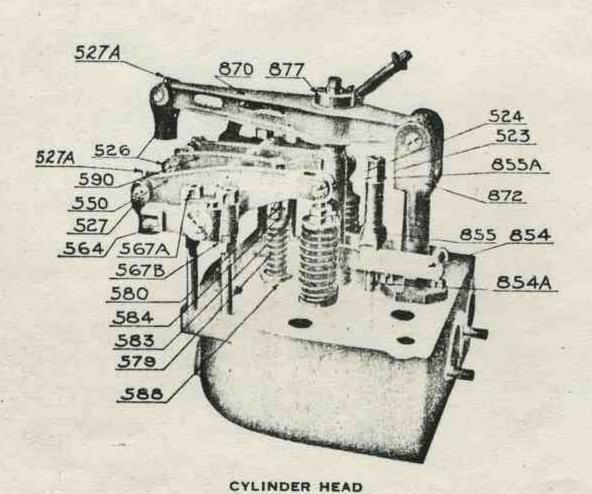
Draw fuel from storage tank through a pipe inserted in top of tank. Be sure the lower end of suction pipe is 3 or 4 inches above the bottom of tank. This allows dirt and water to settle to bottom of tank where they can be occasionally drawn off.

An engine supply tank should be provided big enough to hold at least a day's supply of fuel.

Strain the fuel oil when transferring it from storage tanks to engine supply tank. A good strainer can be made from a five-gallon can by cutting off the top and inserting therein a conical strainer made from 150-inch bronze gauze. Cut the gauze into 4 triangles and solder the upper edges to the top of the can. Have the lower point of the gauze about 2 inches above the bottom of the can. Punch a hole in the bottom of the can and solder on a piece of 1½" brass pipe, which can be inserted in engine supply tank filling hole. Be sure to use a new bright clean can free from rust or dirt. Keep the strainer away from dust and dirt when not in use and wash out frequently.

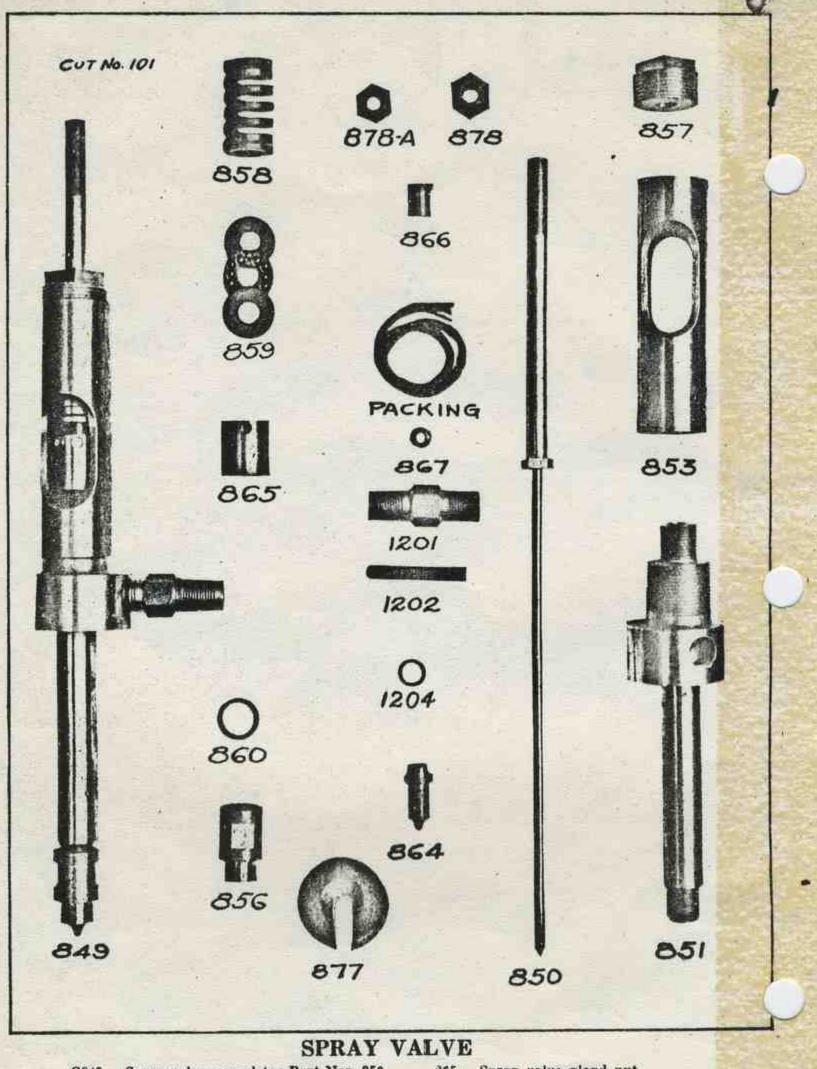






Cylinder Head

G500	Cylinder head complete as-	G550	Exhaust rocker with roller
	sembly (as shown on page		and pins. 550, 550A, 523,
	. 12) except spray valve G849	FEOA	524, G527A.
	and horse shoe collar 877,	550A	Exhaust rocker bushings (set
	the following parts are in-	-01	of 2)
THE REAL PROPERTY.	cluded, Nos. 500, 510, 512,	564	Rocker shaft, connecting link
James and	513, 514, 514A, G520, G550,		(female).
	564, 564B, 565, 566, 567,	564A	Rocker shaft connecting link
300000000000000000000000000000000000000	567A, 567B, 579, 580, 582,		(male).
	583, 584, 585, 586, 588, G590,	564B	Rocker shaft connecting link
	764, 854, 854A, 855, 855A,	V 72.22	pin.
	G870, 872, 873.	565	Rocker shaft.
	Specify whether handle or	566	Rocker shaft bearing (state:
	male connecting link is		with or without wings).
	wanted with head. The fol-	567	Rocker shaft bearing stud.
	lowing parts are also in-	567A	Rocker shaft bearing stud
	cluded if used: 515, 516, 517.		nut; top.
GA500	Cylinder head with exhaust,	567B	Rocker shaft bearing stud
	inlet, air starting valves	THE RESERVE	lock nut.
	and springs only. Part Nos.	G570	Air starting handle, complete:
	500, 510, 512, 513, 514, 514A,		Part Nos. 570, 571, 571A,
	564, 579, 580, 582, 583, 584.		572, 573.
	585, 586, 588.	570	Air starting handle.
500	Cylinder head only with studs	571	Air starting handle pawl.
	567 and 764.	571A	Air starting handle pawl screw
501	Cylinder head stud.	572	Air starting handle pawl spring
501A		573	Air starting handle sector.
	for lifting head.	G580	Air starting valve complete.
502	Cylinder head gasket.		Cage type-Parts No. 580, 581,
510	Valve, inlet or exhaust. (Speci-		581A, 582, 582A, 584, 585,
	fy whether inlet or exhaust)		585A, 586. (See page 19 for
512	Valve stem bushing, inlet or		drawing.)
	exhaust.	G590	Air starting rocker, with roller
513	Valve spring, inlet or exhaust.		and pins. 523, 524, G527A.
514	Valve spring bushing, inlet or		(590A when used).
	exhaust—top.	590A	Air starting rocker bushings
514A			(set of 2).
14,000	exhaust-bottom.	764	Exhaust elbow stud.
515	Valve cage, inlet or exhaust.	G849	Spray valve complete (see
516	Valve cage stud.		page 14 for parts).
517	Valve cage flange.	854	Spray valve clamp.
519	Exhaust or inlet valve cage	854A	Spray valve clamp stud.
	ring.	855	Spray valve clamp bridge.
G520	Inlet rocker, with roller and	855A	Spray valve clamp bridge nut.
	Pin. Parts 520, 520A, 523,	G870	Spray valve rocker.
	524, G527A.		Spray valve rocker pin No.
520A			527.
523	Rocker roller, for inlet, ex-		Spray valve rocker fork No
	haust or air.		526.
524	Rocker roller pin, for inlet,		Spray valve rocker fork pin
APPENDED TO	exhaust or air.	The same of	lock No. 527A.
526	Push rod fork, for inlet, ex-	872	Spray valve rocker stand.
TEL POR	haust, air or fuel.	873	Spray valve rocker stand pin.
527	Fork pin, inlet, exhaust, air	877	Spray valve horse shoe col-
al la l	or fuel.	58	lar.
G527A	Fork pin, ball lock.		
	The state of the s		



G849	Spray valve, complete; Part Nos. 850,	865	Spray valve gland nut
	851, 853, 856, 857, 858, 859, 860,	866	Spray valve gland
	864, 865, 866, 867, 978, 878A, 1201,	867	Spray valve packing seat
	1202, 1204	877	Spray valve horseshoe collar
850	Spray valve stem	878	Spray valve stem nut
851	Spray valve body	878A	Spray valve stem lock nut
858	Spray valve spring casing	895	Spray valve testing outfit
856	Spray valve seat nut	896	Spray valve cleaning needles
857	Spray valve spring plug	897	Spray valve tip or nozzle remover
858	Spray valve spring	898	Spray valve tip cleaner plunger
859	Spray valve spring ball bearing	1201	Fuel strainer body
860	Spray valve gasket	1202	Fuel strainer stem
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Spray Valve

FUNCTION:

The function of the spray valve is to admit the fuel to the cylinders at the proper time and at the same time deliver the fuel oil in the form of a very finely atomized spray so that the fuel is readily ignited and properly burned within the cylinder. The spray valve is located in the center of the cylinder head and extends clear through the cylinder head, with the spray valve tip projecting slightly below the bottom of the cylinder head. It is provided with a needle valve held in position by a strong spring, the tension of which is slightly adjustable. The fuel oil is admitted below the stuffing box at a pressure of anywhere from 1000 to 3500 pounds per square inch (depending on the adjustment of the pressure regulating valve). This spray valve is held in position by a clamp arrangement having a single stud, thus making it easily removable for inspection or cleaning. It is operated by a lifting rocker arm which lifts the valve approximately, 2/100 of an inch when the engine is pulling full load. The governor controls the lift of these valves so that when the engine is running idle the lift is very little and only sufficient to admit enough fuel to turn over the engine. As the load increases the governor causes these valves to raise more and consequently admit additional fuel in direct proportion to the power developed.

SPRAY VALVE TIPS

Sizes used differ according to size of engine (bore and stroke) and are as follows:

61/2 inch bore 81/2 inch stroke use size 8 tips with 5 holes. 7 inch bore 10 inch stroke use size 8 tips with 5 holes. 7½ inch bore 10½ inch stroke use size 8 tips with 5 holes. 81/2 inch bore 12 inch stroke use size 8 tips with 5 holes. inch stroke use size 10 tips with 5 holes. 9 inch bore 12 inch stroke use size 10 tips with 5 holes. 9½ inch bore 13 10 inch bore 13 inch stroke use size 11 tips with 5 holes. inch stroke use size 12 tips with 5 holes. 11½ inch bore 15 inch bore 16 12 inch stroke use size 14 tips with 5 holes. 14 inch bore 18 inch stroke use size 14 tips with 5 holes.

CARE AND MAINTENANCE:

To insure proper functioning of the spray valve it is paramount that it be kept clean and the holes in the tips be absolutely clear. The packing on the stem should be screwed up just tight enough to hold the pressure, for if too tight it will cause excessive wear on the valve stem, thereby causing the valve to work in a very sluggish manner or to remain open allowing too much fuel to enter the cylinder. When such a condition exists the cylinder relief valve will begin to blow off due to the excessive pressure which arises in the cylinder under such conditions.

If any of the tips become clogged with dirt or any other matter the fuel is unable to enter the combustion chamber in its usual manner and will cause a reduction in the revolutions of the engine, also in power. This condition is more apt to occur in a new installation due to dirt

that is in the piping and the tips will have to be cleaned until this disappears or is worked out. If clean pure fuel is used the tips will only have to be cleaned on very few occasions.

When the valves have been disassembled to be cleaned care must be taken to readjust them properly when placing them in the cylinder heads. The maximum lift of the spray valve spindle should never be more than 2/100 of an inch in any case. Excessive lift will only be an unnecessary waste of the fuel while not sufficient lift will cause the lifters and cams to make excessive and unnecessary noise. The screw strainer on the side of the spray valve should be checked to see that it is not striking the spray valve stem and is also screwed in the strainer far enough to miss the nipple part of the union.

ADJUSTING AND TIMING OF SPRAY VALVES:

Put the governor control handle in full speed position; place the engine starting lever in running position. Close all isolating valves on fuel rail. Open all cylinder relief or snifter valves. Bar over flywheel in running direction until pointer on engine frame corresponds to a few degrees before mark on flywheel showing opening of spray valves. Then open isolating valve corresponding to cylinder you are timing. Be sure that the piston in that particular cylinder is nearing the top of compression stroke, which is easily ascertained by the fact that both inlet and exhaust valves remain closed on this stroke. CAUTION: NEVER ATTEMPT TO TURN THE FLYWHEEL WITH THE BAR UNLESS ALL CYLINDER SNIFTER VALVES ARE ENTIRELY OPENED.

Pump up fuel pressure by hand fuel pump to about two thousand (2000) pounds pressure. Then continue barring the flywheel over slowly until the pointer on the engine frame registers with mark on flywheel showing opening of the spray valve. (Referring to diagram A on page 21.) If at that point the fuel pressure on gauge does not release, then loosen jam nut under the push rod fork for that particular spray valve, and adjust the push rod by screwing it in or out as the case may be, until the pressure is released at that particular point. If the pressure is released before the flywheel is turned to the proper point indicated on the flywheel, then the rod is too long and must be shortened by screwing it into the fork slightly; and if the pressure does not release when the flywheel has been turned to the proper point as above referred to, then the rod should be unscrewed and thereby lengthened until it releases at the exact point.

When this has been accomplished lock the jam nut tight under the fork, turn the flywheel back again to see if the tightening of jam nut has slightly altered the timing, in which case it must be loosened again, and the rod screwed into the fork slightly farther, and the jam nut tightened again. Turn the flywheel backwards again a few degrees, then turn ahead very slowly to see that the pressure releases exactly at the desired point.

It will be necessary to pump up pressure with the hand pump each time.

In order to ascertain the time of closing the spray valve, turn the engine ahead until the mark on flywheel showing closing of spray valve

is past the pointer. Then pump up pressure, bar the flywheel backwards again until the pressure is released. This will determine the

exact point on the flywheel where the spray valve closes.

If this varies a little from 18 degrees DO NOT change push rod, as time of opening of the valve is most important. If it closes earlier than 15 degrees or later than 20 degrees then the cam on the cam shaft must be turned away from you to make the closing come later and toward you to make the closing come earlier. After moving the cam the push rod must again be adjusted. Changing the location of the cam should only be necessary at very infrequent intervals.

Once a week the timing should be checked over and the valves

corrected.

Proceed in this same manner with the spray valves on each cylinder on which valve timing needs checking up. It is important that the valve timing of the spray valves be correct on all cylinders in order that they all may be working equal when the engine is running.

In timing the spray valves a certain amount of oil enters the cylinder when the fuel pressure is released. In order to clear the cylinders of the excess oil ALWAYS TURN THE ENGINE OVER ON AIR WITH THE SNIFTER VALVES OPEN, being sure of course that the isolating

valves are closed, before starting on fucl.

If black smoke issues from the exhaust pipe ascertain which cylinder is smoking, the cause of the smoke being either due to too much fuel entering that cylinder, or that some of the spray tips are partly plugged up, spreading the fuel unequally into the cylinder, or admiting so little fuel to some of the cylinders that an excessive load is put on some one individual cylinder; or, that the spray valve may be leaking either at the valve seat or at the joint between spray tip and spray valve body. It is of utmost importance that the spray valve be absolutely tight so that no fuel can enter the cylinder at any other time excepting during the period when the valve is opened by the spray cam, which should be at all times kept in proper adjustment.

Whenever a spray valve is removed from the head for the purpose of cleaning, grinding, or adjusting, then upon replacing the spray valve the timing of that spray valve must be checked up again as a slight

alteration may be caused by the replacement.

SPRAY VALVE TEST CLAMP:

There is a connection located on the high pressure rail for the purpose of testing out the spray valves after cleaning them, to see that the holes are clean.

Two tapped holes are located directly beneath this connection on the center frame to which a stand is attached for holding the spray valve, and also has a lever for lifting the valve in the same manner as the rocker.

HOW TO USE TEST CLAMP:

First close all the isolating valves on the fuel rail. Then after spray valve has been fastened in the stand attached to the centerframe connect the spray valve, with its connection, to the connection on the fuel rail. After all connections have been inspected and made tight, pump up the fuel pressure by the hand priming pump. Then open the valve by hitting the end of the test stand handle with a quick short tap. Watch closely to ascertain if a fine spray of fuel comes out of EACH separate hole in the tip.

When assembling the spray valve make sure that all of the parts have been put together tight and DO NOT LEAK, for a leaky spray valve is one of the greatest reasons why the engine will not run prop-

erly.

Care should be taken not to forget to close the isolating valves on all of the cylinder heads for if they are left open a large amount of oil will gather on the top of the piston head and when the engine is started is liable to cause trouble, due to the excessive oil being compressed in the cylinder.

TAKE SPECIAL NOTICE

1. A SMOKY ENGINE OR LOSS OF POWER IS DUE TO:

Fuel Valve: Leaks, stem is clogged, nozzle is worn out, tension on spring too loose, or not timed properly, or all of the cylinders are not firing properly.

Exhaust Valves: Leaks, stem stuck in guide or not timed properly.

Intake Valve: Leaks, stem stuck in guide or not timed properly.

Loss of Compression: Leaky cylinder relief valve, cylinder head gasket leaking, too low a piston (extra clearance due to wear and adjustment of connecting rod brasses and main bearings), change to a higher altitude or piston rings stuck in their grooves.

Too low fuel oil pressure. Engine running at too slow a speed.

Too much lost motion in governor parts. Wrong kind of fuel oil.

2. BE SURE TO:

Oil all moving parts with regularity. Watch lubricating oil pres-

Squirt a small amount of kerosene mixed with lubricating oil (3 parts kerosene, 1 part lubricating oil) on each exhaust and intake valve stem previous to each time engine is started. This is not necessary if the engine has just been stopped and is started soon again.

Always slow engine down while idling.

Keep correct amount of lubricating oil in sump or supply tank.

Change lubricating oil when necessary (usually from 250 to 300 hours engine running time). Clean oil strainer every two weeks oftener if required.

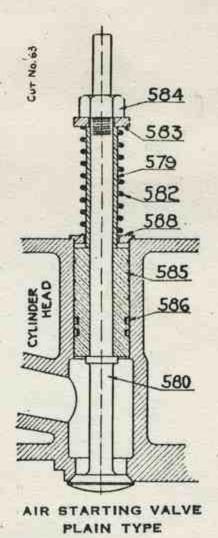
Draw water off bottom fuel oil tank and strainers as often as required to keep water out of engine. Always keep the engine clean.

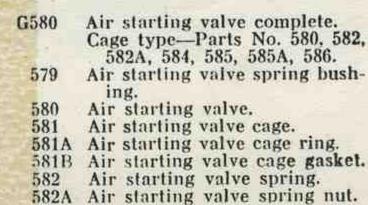
Be sure to examine various parts of the engine while it is running to make sure that none of the parts have become overheated. This is also a good test to see if the circulating water is circulating properly and the lubricating oil still retains its lubricating value.

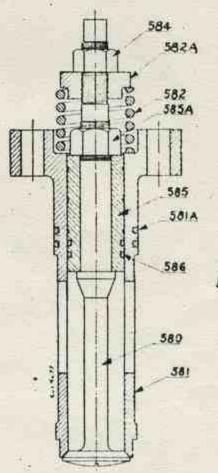
TIMING OF THE AIR STARTING VALVE:

To time the air starting valves, first open all of the cylinder snifter valves, then turn the flywheel in the direction of rotation until top cen-

Air Starting Valve







AIR STARTING VALVE

583	Air starting valve spring wash- er-top.
584	Air starting valve nut.
585	Air starting valve balance bush-
585A	Air starting valve balance bush- ing nut.
586	Air starting valve balance bush-
588	ing ring. Air starting valve spring wash-

er-bottom.

ter of No. 1 cylinder is opposite the pointer attached to the centerframe. Place the air starting valve handle in the starting position as shown in Figure A, page 20. Now adjust the starting valve push rod tight enough so that there is no play between the starting valve stem and the rocker arm roller (or button), then tighten the jam nut on the push rod. This adjustment will time the air starting valve to open at top center. The closing of the valve is taken care of in the shape of the cam. The above process must be repeated with each cylinder to properly time the air starting valve.

The air starting valve opens every other revolution and must be set when the piston is on that top center when both inlet and exhaust valves are closed.

VALVE TIMING:

Referring to the diagrams (page 21):

Diagram A shows the portion of the revolution relative to top center for the opening and closing of spray valves. Remember that this

valve setting is based on the governor being set at full speed position, which in turn will permit the maximum lift of the spray valve.

Diagram B has reference to the intake valve which admits air to the cylinders during the intake stroke. The diagram shows approximately the time that the intake valve should remain open. When the proper time for opening has been adjusted, which is 5° before top center, then the time of the closing of the intake valve will take care of itself, the same as the exhaust valve.

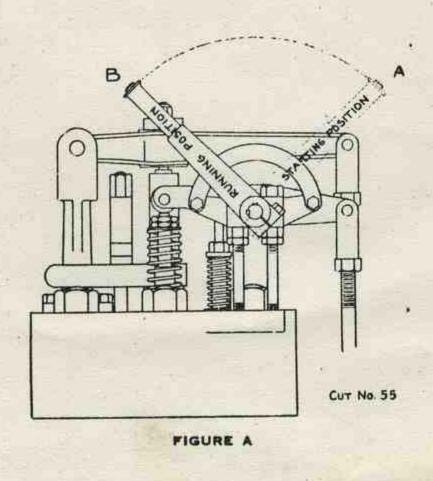
Diagram C—Exhaust valve—shows approximately the period that the exhaust valve should remain open. In timing the exhaust valve only pay attention to the closing time. If the cams are so set as to close the exhaust valve at approximately the correct position, which is 5° past top center, as shown on diagram, then the opening time will take care of itself, due to the size of the lobe of the cam.

It is not essential that the air intake and exhaust valve settings should be exactly on the degree as they may vary two or three degrees either way without interfering with either power, economy or running condition of the engine.

In adjusting the valve setting at any time do so by lengthening or shortening the valve lifter rods by screwing them in or out of the rocker fork which is attached to the outer end of the rockers.

ADJUSTMENT AND TIMING OF ALL THE VALVES:

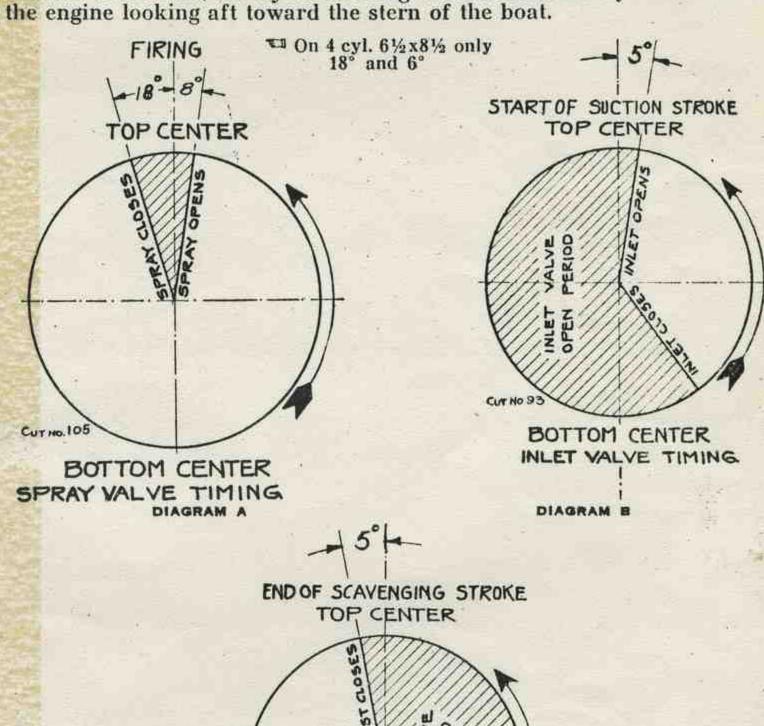
When an engine is shipped from the factory all of the valve timings are adjusted properly for running. If, however, for any reason the engine has been partly dismantled during installation or shipment, or if some one has interfered with the valves settings, (while crating or shipping) then it may become necessary to retime the valves before starting the engine. The most important valve to have correctly timed is the spray or fuel valve, located in the center of the cyl'nder head.



FIRING ORDER OF THE ENGINES:

Number ONE cylinder is the one located next to the flywheel (except on a special engine, or such where the flywheel is placed near the reverse gear and thrust bearing). The firing order for the three cylinder engine is 1-2-3, for the four cylinder engine is 1-2-4-3, and for the six cylinder engine is 1-5-3-6-2-4.

Turning the engine "towards you" means turning the flywheel counter clock wise, the flywheel being viewed from the flywheel end of



BOTTOM CENTER EXHAUST, VALVE TIMING

DIAGRAM C VALVE SETTING DIAGRAMS

CUT NO 94

Fuel Relief Valve--- Marine Type

G1230	Relief Valve, complete: Part Nos. 1117, 1118, 1124, 1125,	1240	Relief valve handle plug, lower.
	1230, 1231, 1232, 1233, 1234,	1241	Relief valve cap nut.
	1236, 1237, 1238, 1238A, 1239,	1242	Relief valve fuel fitting.
	1240, 1242, 1243, 1244, 1245,	1244	Relief valve handle sector.
	1249.	1245	Relief valve spring plug, top.
1230	Relief valve body.	G1243	Relief valve handle, complete:
1231	Relief valve gland.		Part Nos. 1117, 1118, 1124,
1232	Relief valve stud.		1125, 1243, 1249.
1233	Relief valve stem.	1117	Relief valve handle.
1234	Relief valve seat.	1118	Relief valve pawl.
1235	Relief valve stud collar.	1124	Relief valve pawl spring.
1236	Relief valve spring.	1125	Relief valve pawl spring
1237	Relief valve spring cage.		screw.
1238	Relief valve handle bearing.	1243	Relief valve handle cam.
1238A	Relief valve handle bearing	1247	Relief valve stud collar gas-
120074	pin.	1247	ket.
1239	Relief valve handle adjusting	1248	Relief valve stud gasket.
	screw.	1249	Relief valve handle screw.

The fuel relief, is a spring loaded valve connected directly to the high pressure fuel pumps by means of steel tubing, and is equipped with an adjustable spring tension. The purpose of this valve is to regulate the pressure of fuel delivered to the spray nozzles. It is essential that the pressure be kept to about 3800 pounds when the engine is working under normal loads. An increased pressure causes an excess amount of oil to enter the cylinder, thereby increasing the heat. The normal initial temperature at the time of ignition when the engine is developing its rated horse power is approximately 1200 degrees Fahrenheit. Additional fuel pressure makes it possible to raise this temperature up to 2400 degrees Fahrenheit.

The increase in pressure will increase the power to some extent but not in proportion to the increase in pressure or heat produced.

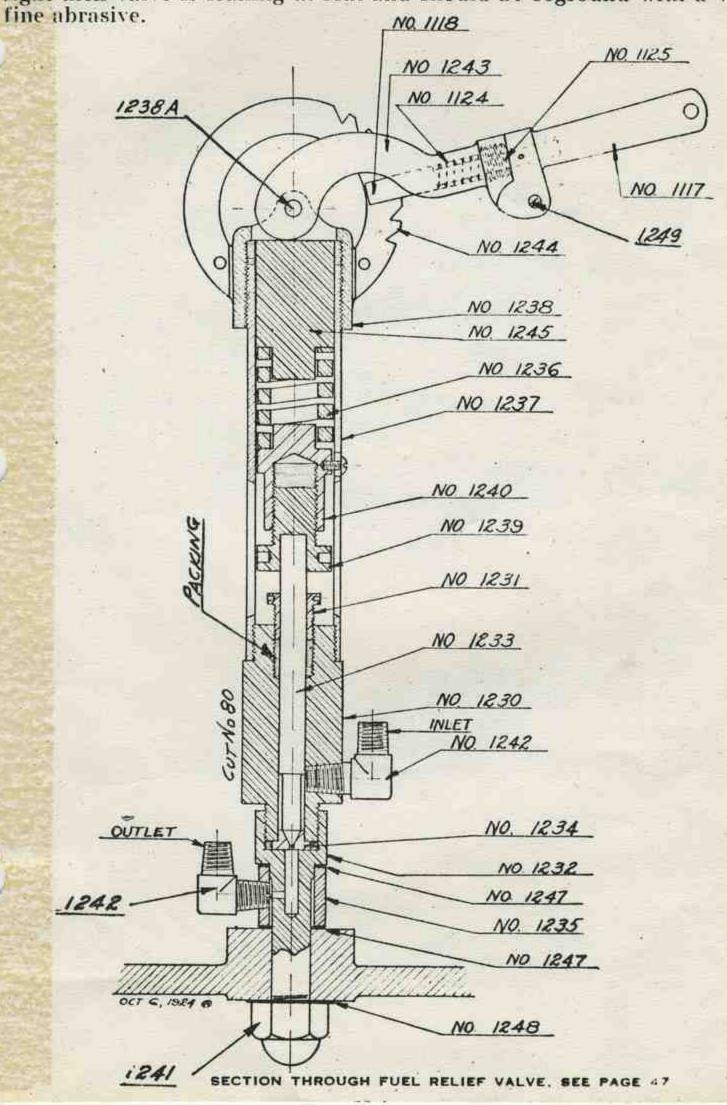
The high momentary temperature has a tendency to distort the metal in the cylinder heads as well as in the top of the piston. The quick changing of temperature from high flash to cold intake period will eventually start minute surface cracks which in time increase in depth as well as in area. This is likely to cause cracked piston heads and cylinder heads.

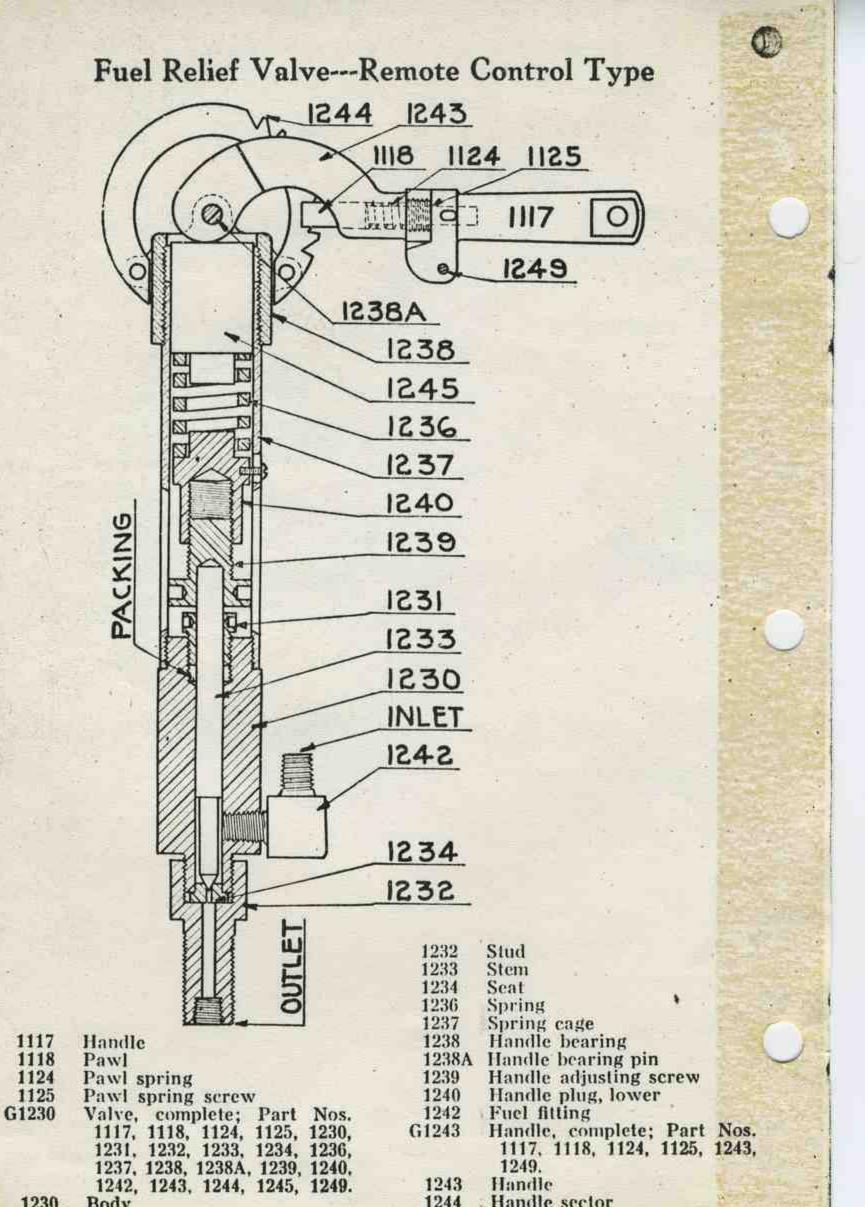
With the increased pressure on the fuel line, more oil is injected into the cylinder than can be used, and consequently it will leave a carbon deposit. This clings to the cylinder head and valves, causing the valves to leak.

It will be readily understood that the higher the pressure on the fuel line the greater will be the quantity of fuel admitted to the cylinders during a given period of spray valve opening. If the engine is idling, the pressure should be reduced to about 1000 pounds. The altering of pressure is accomplished by manipulating the handle on top of the fuel relief valve.

When the engine has been adjusted to a certain pressure of fuel suitable for a fixed load, then it is essential that this pressure be maintained uniformly. This valve is so constructed that any quantity of oil over and above that which goes through the spray nozzles at the fixed pressure will be by-passed and let back into the fuel suction line. In

this way this fuel relief valve automatically maintains the pressure and takes care of the surplus fuel pumped by the high pressure fuel pumps. If the fuel relief valve leaks, make sure that valve stem is working freely and not stuck or sluggish in packing; if this is found to be all right then valve is leaking at seat and should be reground with a very



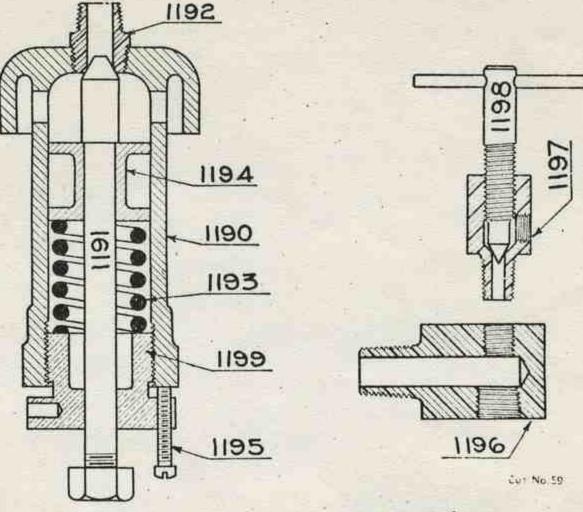


Handle sector Spring plug, top Handle screw

1245 1249

1230 Body 1231 Gland 1231A Gland Packing ring

Cylinder Relief Valves or Safety Valves

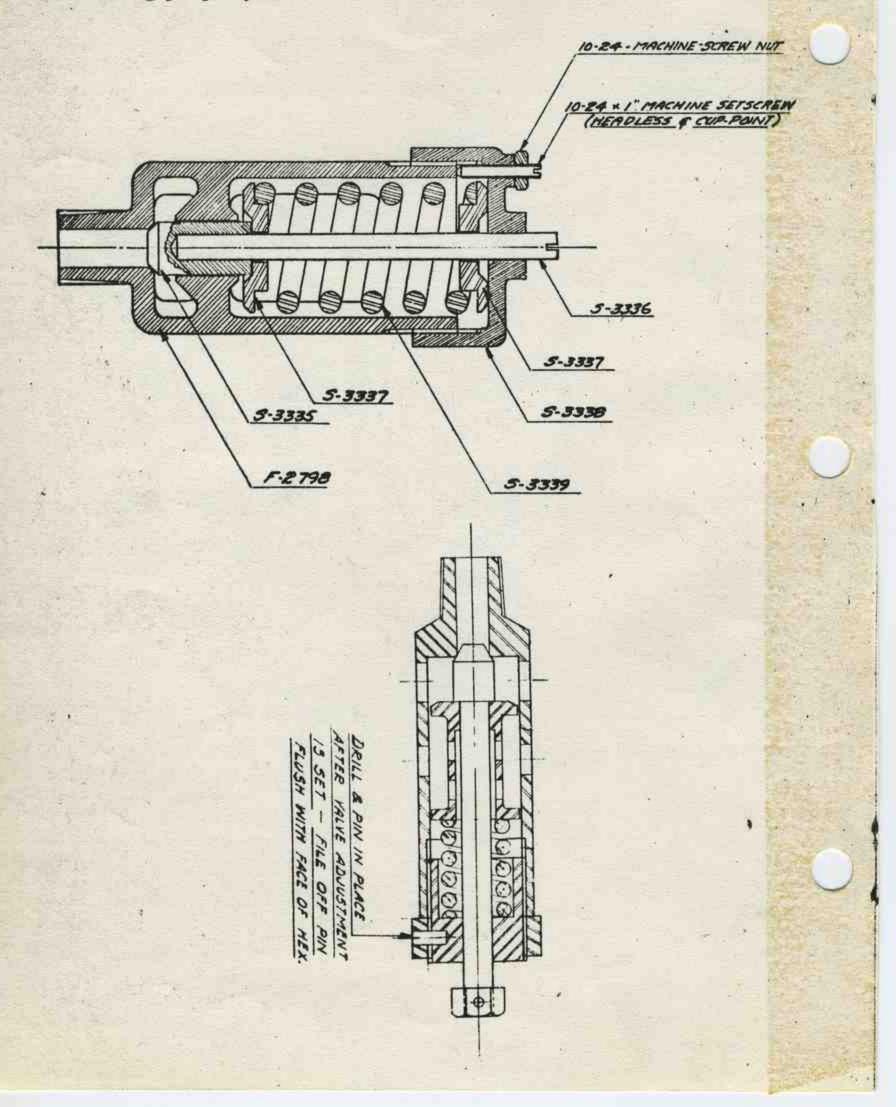


G1190 Cylinder relief valve, com-Cylinder relief valve adjust-1195 plete: Part Nos. 1190, 1191, ing screw lock. 1192, 1193, 1194, 1195, 1199. Cylinder relief valve cylinder 1196 1190 Cylinder relief valve body. Cylinder relief valve adjust-1191 1199 Cylinder relief valve stem. 1192 Cylinder relief valve seat. ing screw. Snifter valve, complete: Part 1193 G1197 Cylinder relief valve spring. Nos. 1197, 1198. 1194 Cylinder relief valve spring 1197 Snifter valve body. collar. 1198 Snifter valve stem.

Three types of cylinder relief valves are used as shown in accompanying cuts.

↑HE function of the cylinder relief valve is to safeguard against excessive pressure in case too much fuel has been admitted to any one cylinder or in case fuel is admitted and ignited before the engine obtains sufficient speed, in which case the pressure may rise excessively. These valves are set at the factory at 750 pounds per square inch and should not be tampered with. Excess pressure escapes through these relief valves. It is just as essential to ascertain from time to time that these valves are in working order as it would be to ascertain at regular intervals that a safety valve on a boiler is in working order. At least once a day they should be tried out. By inserting a screw driver or some such instrument between the nut on the end of the valve stem and the spring tension gland, you can raise the valve from its seat while the engine is running allowing a blast of air or fire to blow from the valve for one or two strokes only. If the valve appears to leak, tap the end of the valve stem lightly with a hammer and at the same time rotating the stem with a wrench on the nut, which will make the valve seat itself again. Care must be taken not to allow these valves to leak

as the heat would soon destroy the valve seat and make them useless. A small steel needle valve, called snifter valve, is attached to these safety valves for the purpose of opening up to see if the cylinders are functioning properly.



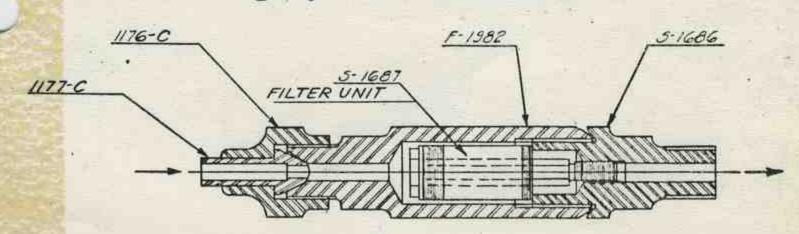
ADJUSTMENT OF SAFETY VALVES:

If while the engine is running the safety valve should blow off (at ordinary running speed) and the spray valve is known to be tight and not sluggish, then it indicates that the safety valve spring is weak or seat of valve is pitted.

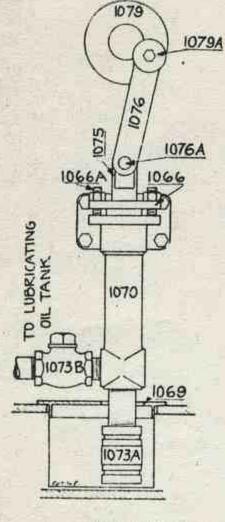
To remedy this, tap very lightly the end of the valve stem with a hammer and with a wrench on valve nut turn valve slowly while tapping. This will cause the valve to reseat itself; if blowing still continues increase tension of the valve spring by turning the adjusting nut until sufficient tension has been exerted on the valve spring to cause the valve to cease from blowing.

Don't tighten up too much or there is danger of setting up the valve spring beyond the safety point. DO NOT TIGHTEN THE SPRING TO OVERCOME VALVE LEAKAGE. If it becomes necessary to regrind the valve to overcome leakage, be sure to get the valve spring adjusting screw back in the same position so as to maintain a pressure of 750 pounds.

Spray Valve Strainers



The Spray Valve Strainer is inserted in the fuel line directly before the spray valve. The function of the strainer is to remove any fine particles of sediment that may have passed through the main fuel strainers. The cleaning of the strainer's element should be required at very infrequent intervals. When it is necessary, the strainer should be removed from the spray valve and taken apart by unscrewing the two ends. The strainer element consists of a unit of laminated disks held together with a screw and nut. Wash the element off in gasoline before loosening the clamp screw. After all the dirt has been removed from the outside of element loosen the clamp screw about an eighth of an inch and rinse it out well in clean gasoline. Then thoroughly clean out the interior of the strainer body and reassemble. Be sure that all particles of dirt have been removed before reassembling, or clogging of spray valve tips will result.



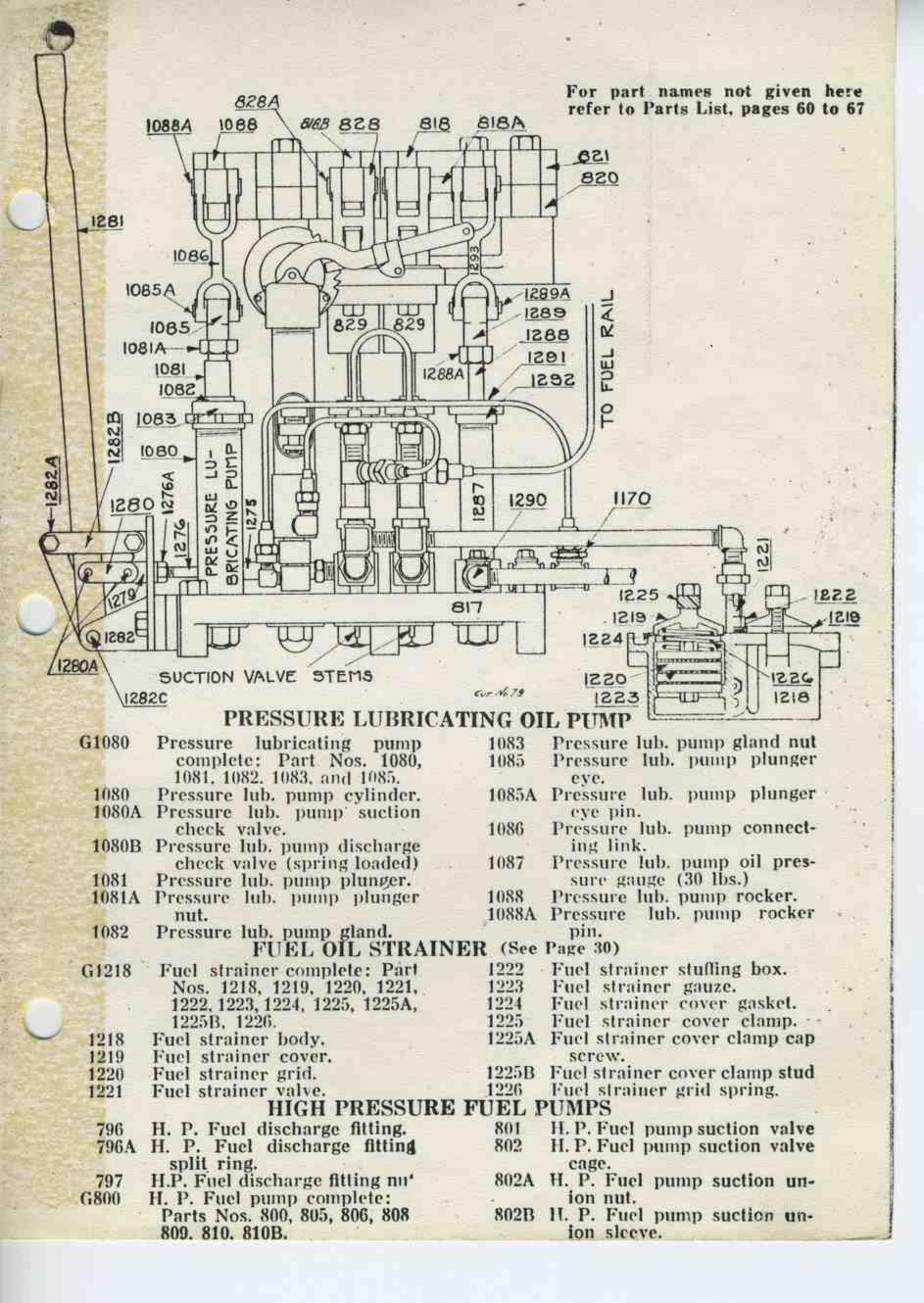
Lubricating Oil Sump Pump

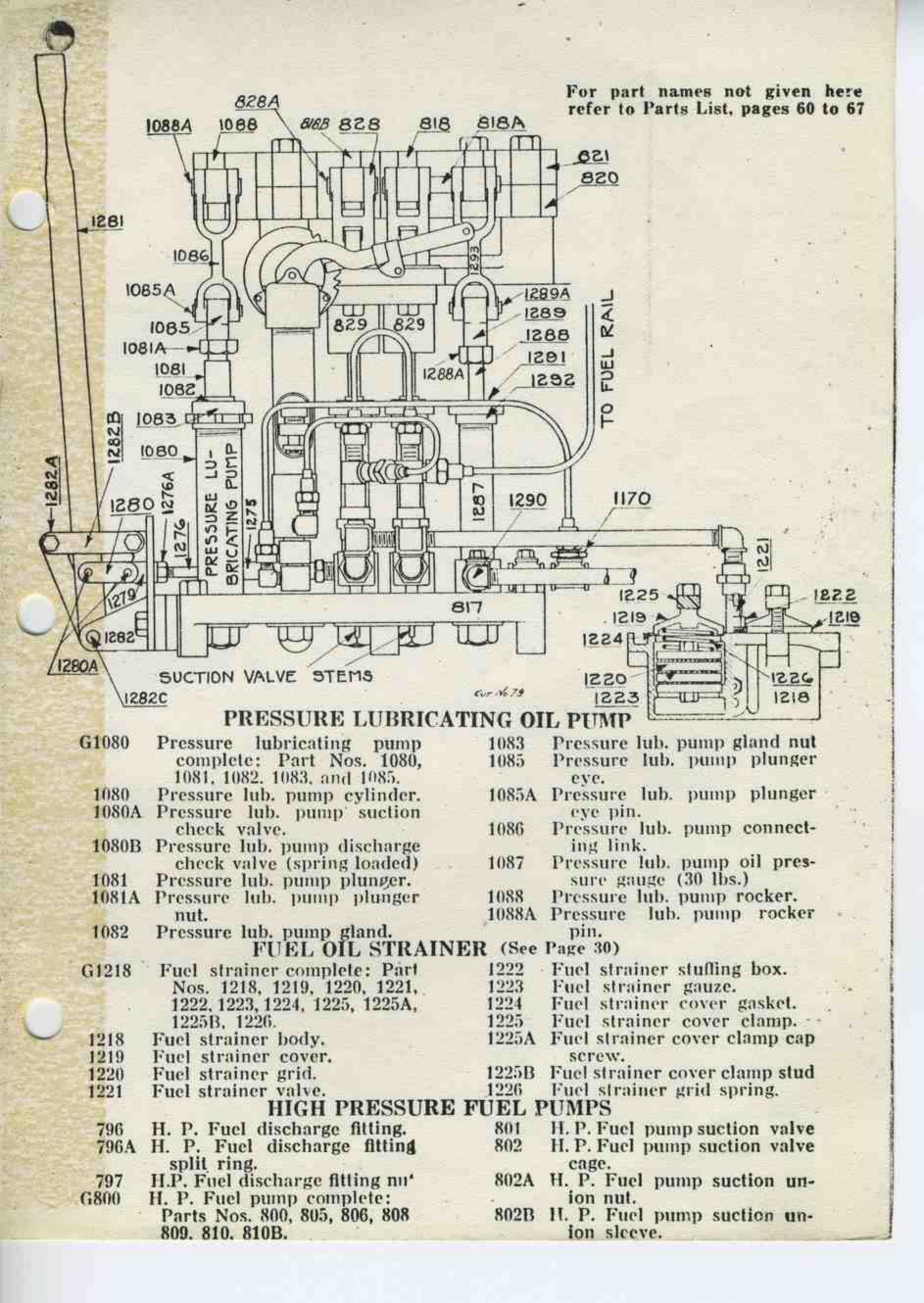
Sump pump gland. 1066 1066A Sump pump gland stud. 1069 Sump pump strainer. Sump pump complete: 1070, 1066, 1066A, 1075, G1070 and packing. 1070 Sump pump body. 1073A Sump pump check valve; inlet. 1073B Sump pump check valve; outlet. Sump pump plunger. 1075 1076 Sump pump connecting rod. 1076A Sump pump connecting rod pin. 1079 Sump pump crank disc. 1079A Sump pump crank washer. Some models use Rocker No. 1088 to drive sump pump in place of crank No. 1079, oth-

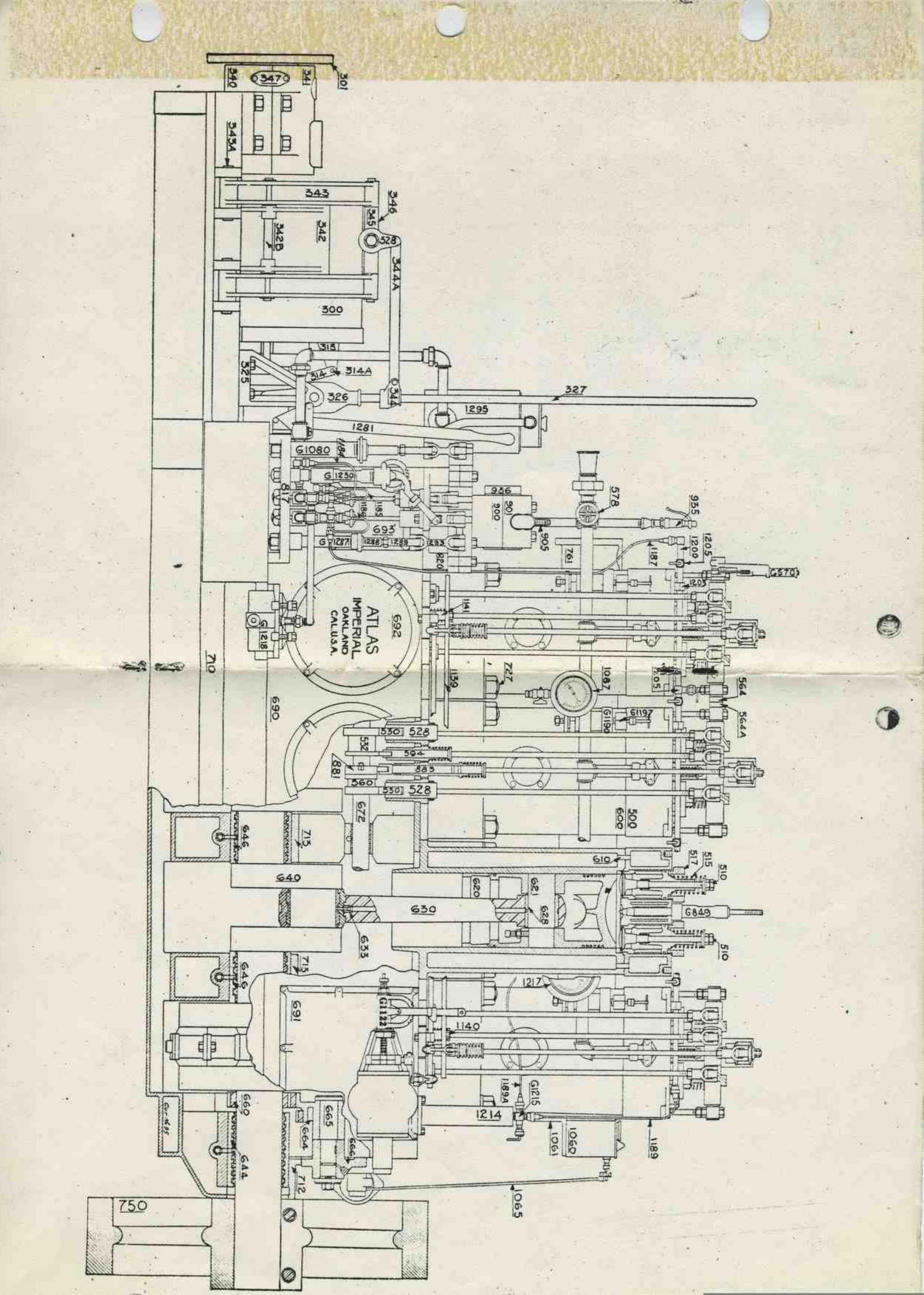
erwise same part numbers apply.

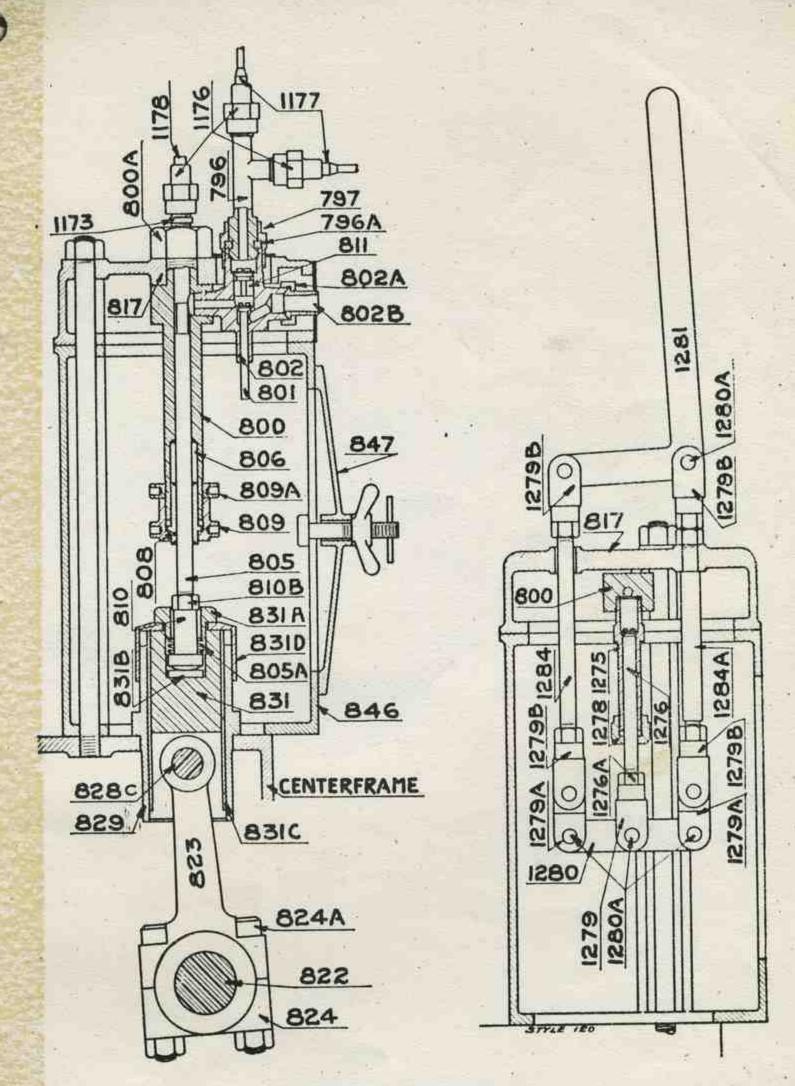
SUMP PUMP

THE lubricating oil sump pump is located on the end on the center-frame and driven by means of a disc crank on the end of the cam shaft. The suction pipe extends down into the lubricating oil sump in the engine base drawing the lubricating oil through a sump strainer and maintaining the proper level of lubricating oil in the sump. The discharge from the pump is conducted to the lubricating oil service tank. Here the oil passes through another strainer settling into the bottom of the tank from which it is drawn by the high pressure lubricating oil pump.









FUEL PUMP

Fuel Pump

	796	High pressure fuel dis-		Fuel numn cross head
	750	charge fitting		Fuel pump cross head
	796A	High pressure fuel dis-	831	guide cap screw
	1001			Fuel pump crosshead
	707	charge fitting split ring	831A	
	797	High pressure fuel dis-	831B	Fuel pump crosshead but-
	000	charge fitting nut	0040	ton
	800	Fuel pump body	831C	Fuel pump crosshead bush-
	800A	Fuel pump body nut		ing
	801	Fuel pump suction valve	831D	Fuel pump crosshead oil
	802	Fuel pump suction valve		guard
		cage	846	Fuel pump housing
	802A	Fuel pump suction union	846A	Fuel pump housing stud
	19.39 (B).0-	nut		Fuel pump housing stud nut
	802B	Fuel pump suction union	847	Fuel pump housing cover
	OULD	sleeve		Fuel pump housing cover
	805			hinge
				The state of the s
	805A	Fuel pump plunger spring		Fuel pump housing cover
	806	Fuel pump packing	****	hinge screw
	808	Fuel pump gland	1173	Fuel pump bleeder connec-
	809	Fuel pump gland nut	0200000	tion
	809A	Fuel pump gland lock nut	1176	High pressure fuel union
	810	Fuel pump plunger head		nut
	810B	Fuel pump plunger nut	1178	High pressure fuel union
	811	Fuel pump discharge valve		sleeve
	817	Fuel pump plate	1178	High pressure fuel union
	200	Fuel pump plate steel cover	-	sleeve, blind end
		(front)	G1274	Priming nump only; Part
		Fuel pump plate steel cover	012/1	Nos. 1275, 1276.
		(top)	1275	
			1276	Priming pump cylinder
		Fuel pump plate steel cover		Priming pump plunger
		screw (large or small)	1276A	Priming pump plunger nut
		Fuel pump plate and hous-	1278	Priming pump cap
		ing slotted cap screw	1279	Priming pump plunger eye
	822	Fuel pump crank	1279A	Priming pump connecting
(1823	Fuel pump connecting rod,		rod link
T)		assembled, Part Nos. 823,	1279B	Priming pump link eye
		824, 824A.		Priming pump link eye nut
	823	Fuel pump connecting rod	1280	Priming pump connecting
	824	Fuel pump connecting rod		rod
	11.00	cap	1280A	Priming pump connecting
	824A	Fuel pump connecting rod	AMOUNT	rod pin
	02471		1281	
		cap bolt		Priming pump handle
		Fuel pump connecting rod	1284	Priming pump handle push
	9000	cap bolt nut	40044	rod
	828C	Fuel pump cross head pin	1284A	Priming pump handle push
	829	Fuel pump cross head	1 1 1 1	rod fulcrum
		guide		# 135 5.32

HIGH PRESSURE FUEL PIPES COMPLETE WITH UNIONS

In ordering fuel pipes or fuel pipe unions, specify outside diameter of pipe. Each pipe complete with parts No. 1176, 1177.

1183 Testing pipe—Fuel rail to test clamp.

1184 Fuel pipe from fuel relief valve discharge to pump suction.

1185 Fuel pipe from high pressure fuel pump discharge to fuel relief valve inlet.

1186 Fuel pipe from connecting pipe between H. P. fuel pumps (discharge).

1187 Fuel pipe from fuel pump to fuel rail.

Fuel pipe from fuel rail to 1188

spray valve. 1189 Fuel pipe from fuel rail to fuel oil receiver.

1189A Fuel pipe from fuel oil receiver to fuel pressure gauge.

1200 Fuel rail with isolating valves. 1170 High pressure union nipple.

1176 High pressure fuel union nut. 1177

High pressure fuel sleeve.

Fuel Oil Strainer

(Parts on Page 48)

HE fuel oil strainer is composed of two compartments or chambers. In each of the chambers is located a built up strainer unit or cartridge, each having three layers of one hundred mesh bronze screen. These cartridges fit snugly into the chamber, and are held in position by means of spiral springs. The spiral springs are held down by the covers which are sealed with copper gaskets.

This strainer unit is provided with a passover valve with ports so constructed that by turning this valve in one position the fuel-oil flows into one compartment of the strainer only, turning it in the other position permits the other strainer to be used. In this way one strainer can be shut off, taken apart and cleaned, while the other is in use,

without interfering with the running of the engine.

It is important that these strainers be cleaned from time to time. The method of cleaning these strainer units or cartridges is by submerging them in a can of gasoline and washing them off thoroughly, remove all sediment in the bottom of the strainer chamber, and wipe all joints clean and free from grit and dirt before again assembling. Grit,

dirt or impurities permitted to enter the fuel pump are apt to create trouble, and for that reason it is important to use as much care as possible in eliminating impurities from the fuel oil.

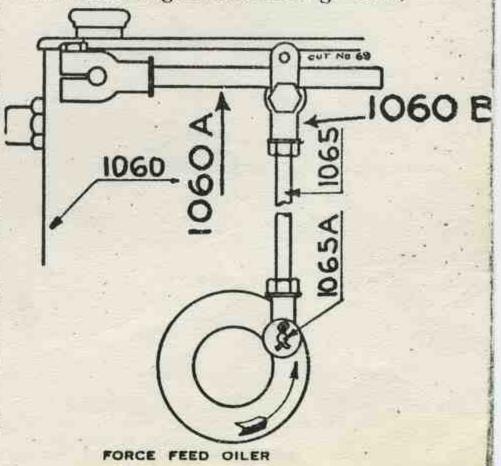
FORCE FEED OILER

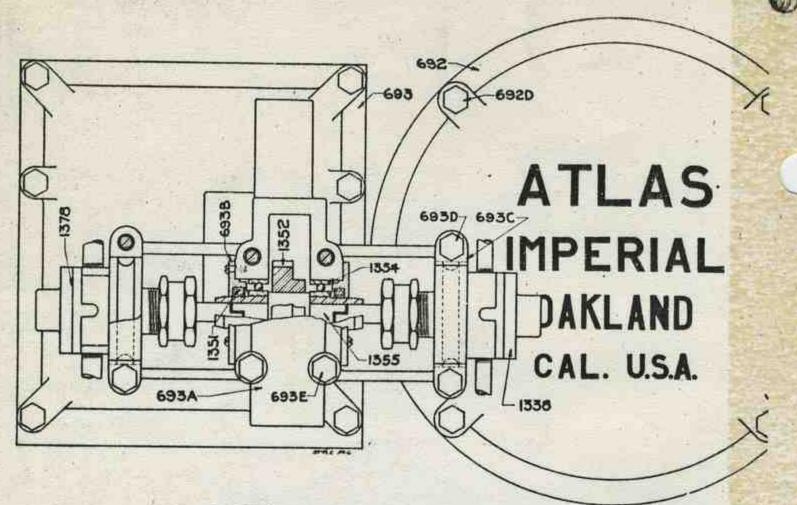
The force feed oiler used for oiling the cylinder walls of the engine is driven off of the cam shaft by means of a crank disc.

Some models lub. drive off water pump connecting rod. Same part numbers apply.

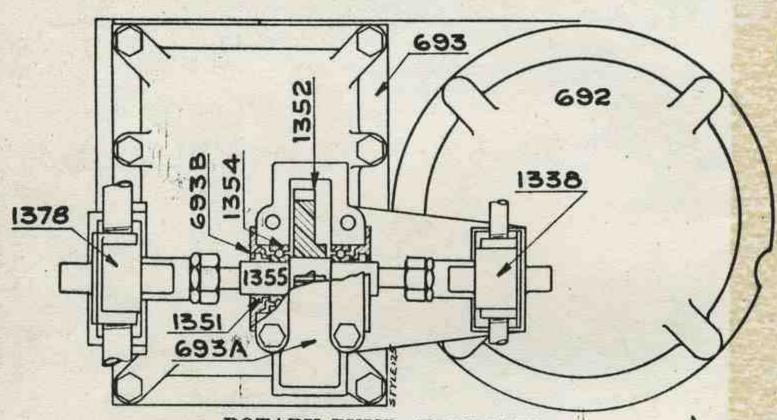
1060 Force feed lubricator (see note) Force feed lubricator drive rod 1065A Force feed lubricator drive rod

pin. Note: When ordering new lubricator, state manufacturer, number of feeds, also type.





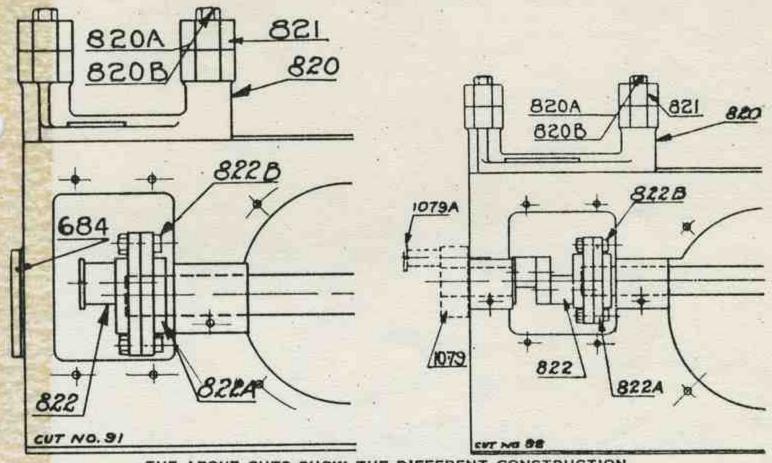
ROTARY PUMP—CLAMP STYLE



ROTARY PUMP-FOOT STYLE

692	Centerframe door (round)	1341	Rotary pump gear and shaft driver for
692A	Centerframe door gasket		1/4" outlet pump
692D	Centerframe cover cap screw	1342	
693	Rotary pump door	1351	Rotary pump felt washer
	Rotary pump door cap screw	1352	Rotary pump gear, driven
6924	Rotary pump door gasket Rotary pump door bearing cap	1353	Rotary pump gear driver on cam shaft
gash	Potons numer door bearing cap		coupling
dead	Rotary pump door ball bearing r		Rotary pump ball bearing
	tainer	1355	Rotary pump drive shaft
1002	Rotary pump door ball bearing r tainer screw	1371	Rotary pump gear and shaft for 1/2" outlet pump. Driver
698C	The same of the sa	1372	Rotary pump gear and shaft driven for
693D			1/4" outlet pump
698E	Rotary pump door cap-cap screw	1378	
1338	Rotary pump, complete; 1/4" outlet	1010	Rotary pump, complete: 1/2" outlet Rotary pump holding down cap screw

FUEL PUMP CRANK SHAFTS

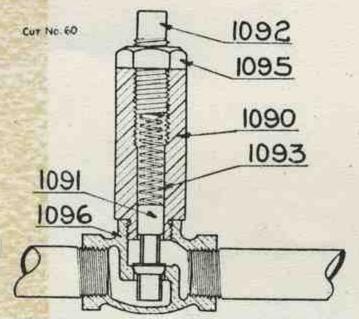


THE ABOVE CUTS SHOW THE DIFFERENT CONSTRUCTION EMPLOYED IN FUEL PUMP CRANK SHAFTS.

THE PART NUMBERS ARE THE SAME IRRESPECTIVE OF THE NUMBER OF CRANKS

684	Centerframe cove	er—en	d of
G820	Fuel pump rocker ing and caps.	shaft	bear-
820A	Fuel pump rocker ing stud (long).	shaft	bear-
820B	Fuel pump rocker ing stud (short).	shaft	bear-
821	Fuel pump rocker ing cap.		bear-

822 Fuel pump crank shaft.
822A Fuel pump crank shaft coupling.
822B Fuel pump crank shaft coupling bolt.
1079 Lubricating sump pump crank disc.
1079A Sump pump crank disc washer



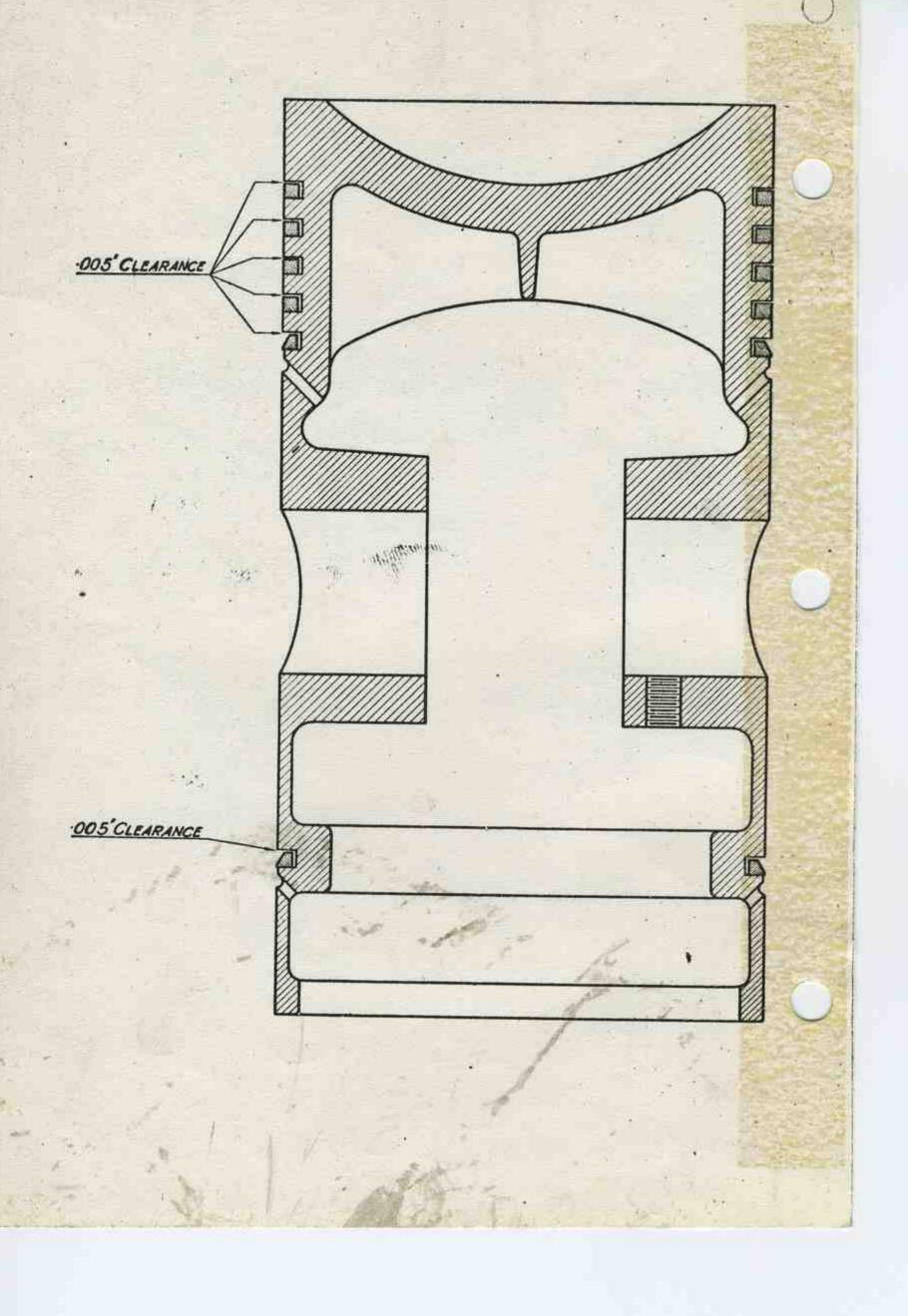
PRESSURE LUBRICATING RELIEF VALVE

			THE PARTY OF	T(OLIVERO)
(1090	Valve, comp 1090, 1091	olete: , 1092	Part , 1093,	Nos. 1095,
1090	1096. Lubricating	relief	valve	body
1091	Lubricating	relief	valve	plun-

PRESSURE LUBRICATING RELIEF VALVE

A pressure lubricating relief valve, used in connection with the pressure lubricating oil pump, is used to maintain a constant and determined pressure of the lubricating oil on its course through the engine. To adjust the valve: first loosen the adjusting screw nut and then tighten or loosen the adjusting screw to suit requirements desired. Then tighten the adjusting screw nut.

1092	Lubricating relief valve ad	-
	justing screw.	
1093	Lubricating relief valve spring	g
1095	Lubricating relief valve ad	
	justing screw nut.	
1096	Lubricating relief valve check	c
	valve and body.	a.



Piston Rings

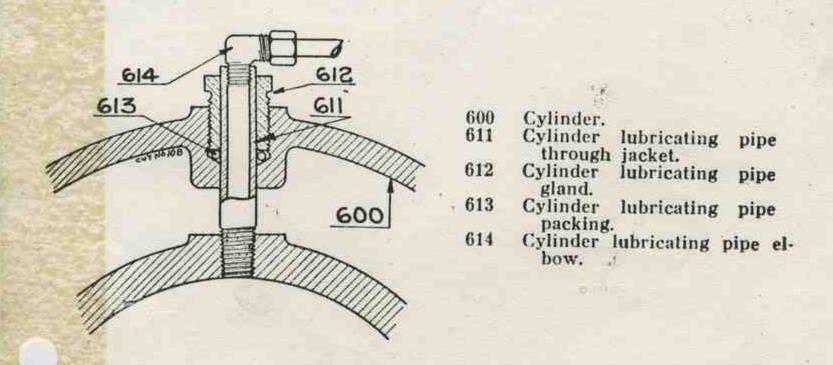
ALL pistons are fitted with six piston rings, five above the pin and one below, as illustrated. All rings are step cut rings. The rings just above the pin and the ring below the pin are oil scraper rings. are should be taken to have these rings properly fitted in the ring grooves as shown.

In fitting new rings a vertical clearance of .005" should be given all rings and it is also important that there be plenty of clearance under the rings and that the sides of the ring grooves be parallel in order to realize the full effect of the rings. If the grooves are worn or tapered it will be necessary to face the sides of the grooves a little wider.

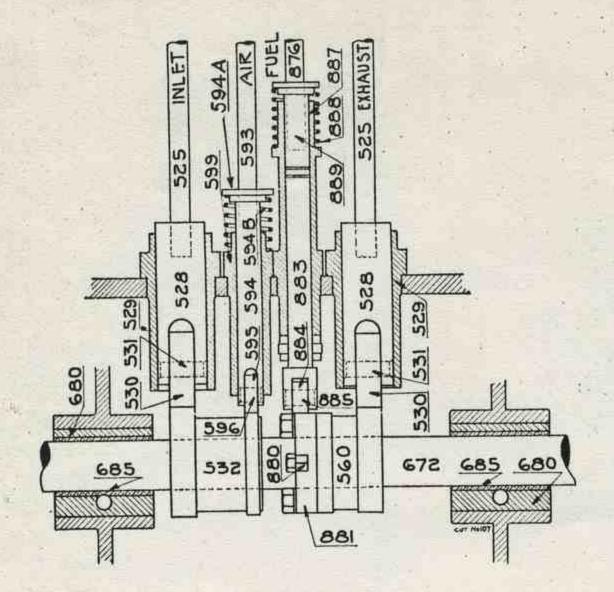
The opening between ends of rings No. 1, 2 and 3 should be not less than .005" per inch of cylinder diameter, while No. 5 and No. 6 ring should be fitted with about .002" per inch of cylinder diameter.

All ring openings should be measured with the rings in the cylinder.

Cylinder Lubricating Pipe



The cylinder lubricating oil pipe shown here for lubricating the piston and cylinder walls of engine requires but three or four drops of oil per minute. It is fed from a force feed oiler through a small copper pipe. A gland and packing keep the water from leaking out of the cylinder jacket.

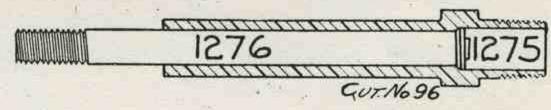


Valve Lifters

525	Push rod, inlet or exhaust.	596	Air starting valve lifter roller
G528	Valve lifter, with roller and	599	Air starting valve lifter guide.
529	Valve lifter guide, inlet or ex-	672 G680	Cam shaft bearing and bushing.
530	Valve lifter roller, for inlet of exhaust.	685 876	Cam shaft bushing. Spray valve rocker push rod.
531	Valve lifter roller pin, for inlel	880 881	Spray valve cam disc.
532	Inlet and air starting cam.	G883	Spray valve lifter, with roller
560	Exhaust cam.	884	and pin. Spray valve lifter roller.
593	Push rod, air starting.	885	Spray valve lifter roller pin.
G594	Air starting valve lifter, with	887	Spray valve lifter guide. Spray valve lifter guide spring.
594A 594B	Air starting valve lifter collar. Air starting valve lifter spring.	888 889	Spray valve push rod socket.
505	Air starting valve lifter roller		

PRIMING PUMPS

A priming pump, No. 1275, connected directly to the high pressure fuel pump and equipped with a hand lever, No. 1281 (See pages 28 and 32). When starting the engine this pump is used for the purpose of exhausting any air from the fuel oil piping and to pump up pressure on the high pressure pipes and manifolds so that when the engine is started the fuel is already at the point of the spray nozzles ready to be admitted to the cylinder as quickly as the spray nozzles are opened by the spray cam.



PRIMING PUMP

G1275	Priming pump, complete: Part Nos. 1275, 1276, 1276A, 1278	1280A 1281	Priming pump plunger pin. Priming pump handle.
The state of	and packing.	1282	Priming pump handle bracket.
1275	Priming pump cylinder.	1282A	Priming pump handle bracket
1276	Priming pump plunger.		spacer.
1276A 1278	Priming pump plunger nut.	1282B	Priming pump handle bracket bar.
1279 1280	Priming pump plunger eye. Priming pump plunger con-	1282C	Priming pump handle bracket lever pin.
	necting rod.		ACTOR PAIN

DAY TANK PUMP SEE PAGE 28 FOR DRAWING OF PUMPS

G1287	Day tank pump, complete: Part Nos. 1287, 1288, 1288A.	1289A	Day tank pump plunger eye pin.
	1289, 1291, 1292.	1290	Day tank pump check valve.
1287	Day tank pump body.	1291	Day tank pump gland.
1288	Day tank pump plunger.	1292	Day tank pump gland nut.
1288A	Day tank pump plunger nut.	1293	Day tank pump connecting
1289	Day tank pump plunger eye.		rod.

High Pressure Fuel Pumps

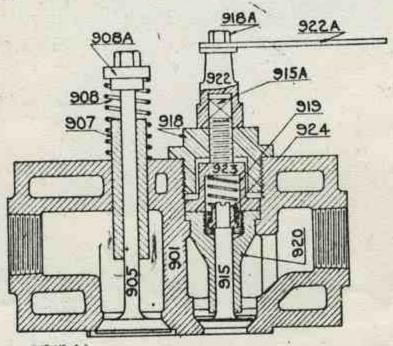
THE fuel pumps are of the plunger type with ½-inch diameter, bardened and ground plungers. These pumps are to pump the fuel up to the required pressure, and for that reason are very closely fitted. It is necessary to use care in packing these plungers. The packing glands should never be screwed down so hard as to make the gland absolutely tight. It is essential that a small amount of fuel oil be allowed to work up through the packing and thereby lubricate the plunger.

A quantity of packing is furnished with each engine. If the packing gland is screwed down too tight the packing will become dry and hard, and thereby cause the plunger to wear. The lower ends of the suction valves extend sufficiently below the valve bodies so that they can be reached by hand, turned around and moved with the fingers. If a valve should be too tight at the stem it can be polished off with fine emery cloth, care being observed not to take off too much. In most cases it can be loosened by moving it up and down and twisting it around with the fingers.

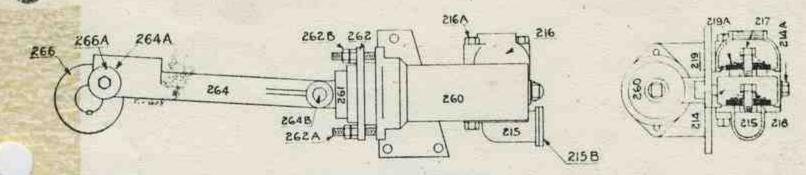
Air Compressor

Purposely no provision has been made for continuous lubrication of this unit. It is intended that the operator should use a squirt can and give this a few drops of oil at the air intake when first starting the engine. After that it receives plenty of oil from the vapor within the base. If a continuous oil drip were used on the air compressor, the valves would carbon up very quickly and there would be danger of the vapor from the lubricating oil igniting, in which case the pipe or tanks would be carried away by an explosion. After the engine has run for a few hours, it will never need any more lubrication on the air compressor except what is received from vapor within the crank case.

918D Air compressor inlet valve Air compressor cylinder with 900 flange gasket. Air compressor discharge valve 919 Air compressor head, complete: G901 Part Nos. 901, 905, 907, 908, spring. Air compressor discharge valve 920 908A, 915, 915A, 918, 918A, 919, 920, 922, 922A, 923, 924. Air compressor discharge valve 922Air compressor head with 907. 901 cap nut. Air compressor cylinder head 902 922A Air compressor discharge valve stud. relief handle. Air compressor inlet valve. 905 Air compressor discharge valve 923 Air compressor grid. 906 guide cap. Air compressor inlet valve 907 Air compressor discharge valve 924 bushing. spring bushing. Air compressor inlet valve 908 Air compressor piston. 925 spring. Air compressor piston ring. 908A Air compressor inlet valve nut. 926 Air compressor piston pin. 927 908B Air compressor inlet valve Air compressor piston pin 928 washer. Air compressor discharge valve bushing. 915 Air compressor piston connect-G929 915A Air compressor discharge valve ing rod; bushed. adjusting screw. Air compressor eccentric strap G930 Air compressor discharge valve 918 (2 halves) and bolts. Air compressor eccentric strap 918A Air compressor discharge valve 932 bolts. relief handle nut. Air compressor eccentric. 933 918B Air compressor inlet valve cap Air compressor relief valve. 935nut stud. Air compressor pass-over pipe. 936 918C Air compressor inlet valve Air compressor gauge; 300 lbs. 937 flange stud.



SEE PAGE 27 FOR DRAWING OF AIR COMPRESSOR



PLUNGER PUMP

G260	Plunger pump complete: Parts 260, 261, 262, 262A, 262B.	
214	Plunger pump valve cage.	
214A	Plunger pump valve cage cen- ter stud.	
215	Plunger pump suction bonnet.	
215A	Plunger pump bonnet copper asbestos gasket.	
215B	Plunger pump inlet flange.	
216	Plunger pump discharge bonnet	
216A	Plunger pump bonnet connect- ing stud.	
217	Plunger pump valve guide stem	8
218	Plunger pump valve spring.	
219	Plunger pump rubber disc valve	
219A	Plunger pump valve bushing.	

GA260 Plunger pump body with 262A.

261 Plunger pump plunger.

262 Plunger pump gland.

262A Plunger pump gland stud.

262B Plunger pump gland stud nut.

G264 Plunger pump connecting rod with 264A.

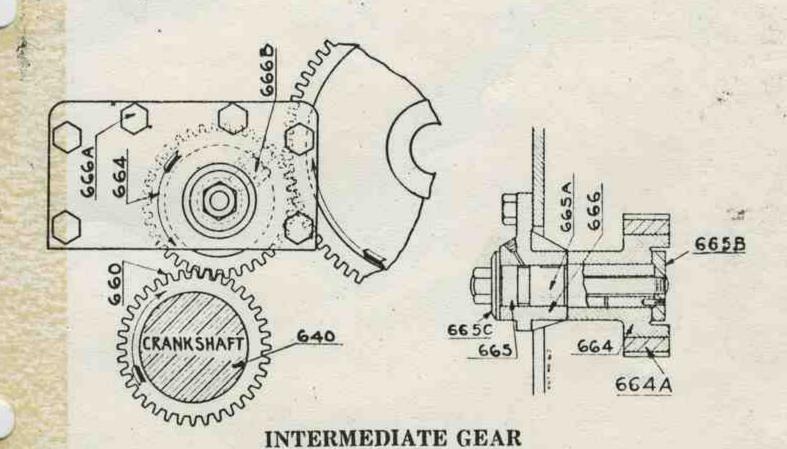
264A Plunger pump connecting rod bushing.

264B Plunger pump connecting rod pin.

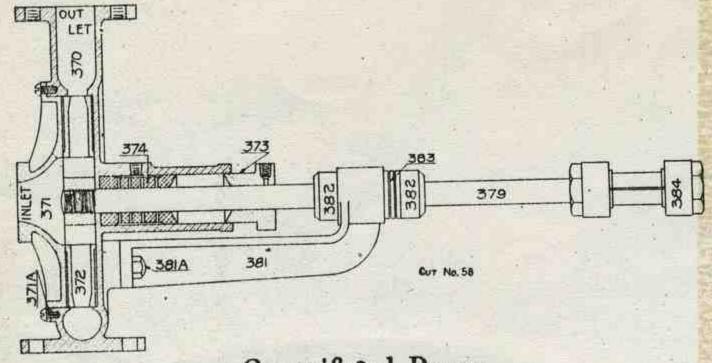
266 Plunger pump connecting rod crank disc.

266A Plunger pump connecting rod

crank disc washer.



640 Crankshaft with 660 and 655. 660 Crankshaft pinion. 664 Intermediate gear (inc. 664A). 665 Intermediate gear pin. 665A Intermediate gear pin bolt. 665B Intermediate gear pin bolt. washer. 665C Intermediate gear pin washer.
666 Intermediate gear bearing.
666A Intermediate gear bearing cap
screws.
666B Intermediate gear bearing lock
set screw.



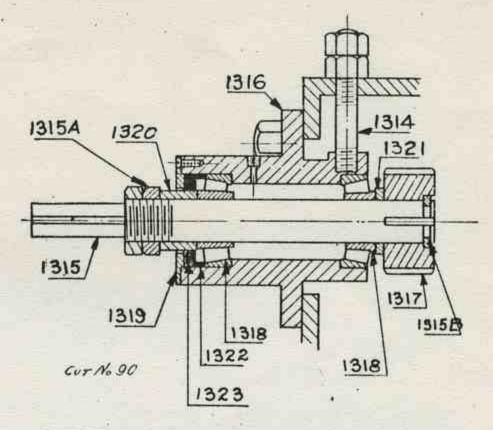
Centrifugal Pump

G370	Centrifugal pump, complete: Part Nos. 370, 371, 371A, 372, 373, 373A, 374, 379, 381, 381A,	373 373A 374 379	Centrifugal pump gland. Centrifugal pump gland stud. Centrifugal pump shaft bushing Centrifugal pump shaft.
emiseus.	382, 383.	381	Centrifugal pump steady bear-
GA370	Centrifugal pump body with	301	ing.
370A	Centrifugal pump outlet flange.	381A	
371	Centrifugal pump cover.	20020	ing cap screw.
371A	Centrifugal pump cover screws.	382	Centrifugal pump collar.
371B		383	Centrifugal pump ball bearing.
The second secon	Centrifugal pump runner with	384	Centrifugal pump coupling.
G372	shaft.		

THE centrifugal pump drive is by means of a silent chain from the crankshaft to the centrifugal pump shaft. With continued use, a slight amount of wear in the links will allow the silent chain to stretch and provision is made for taking up this stretch by means of a chain-adjusting spindle, as shown next page. Occasionally it is well to inspect the chain and take up apparent slackness. This is done by loosening the idler spindle bearing nuts and tightening up on the chain adjusting screws, No. 1314. When the proper tension has been obtained tighten the bearing bracket bolts again and secure them with wire to prevent from becoming loose. Tighten the lock nut on the chain tension adjusting screw. The chain must never be too tight. When properly adjusted there should be sufficient slack in the middle of the span between the pump sprocket and idler sprocket so that it can be lifted up and down with the fingers, about one inch.

Do not allow the chain to become too slack as there is danger of the chain riding over the teeth on the sprocket which would ruin both the sprocket and the chain. Watch the chain tension and take up when

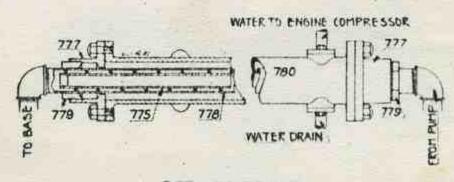
necessary.



CENTRIFUGAL PUMP CHAIN DRIVE

			SENSE.	
Ž	1306	Chain sprocket (driver on	1315	Idler spindle.
		crank shaft).	1315A	Idler spindle lock nut.
	1307	Chain sprocket (driven on	1315B	Idler spindle split ring.
		pump shaft).	1316	Idler spindle bearing housing.
	1308			ther spindle bearing nousing.
		Chain shield.	1317	Idler spindle chain sprocket.
×	1309	Chain shield cap.	1318	Idler spindle roller bearing
	1310	Ball bearing container (blind		(complete).
		end).	1319	Idler spindle steel cover.
	1310A	Ball bearing container (shaft	1319A	Idler spindle steel cover screw
		end).	1320	Idlan animals accretioner screw
	1011		1520	Idler spindle roller bearing
	1311	Driven chain sprocket shaft.		spacer.
	1312	Driven chain sprocket ball	1321	Idler spindle spacer.
		bearing.	1322	Idler spindle spacer washer.
	1313			idici spindie spacer washer.
	THE REPLIES OF THE PARTY OF THE	Silent chain.	1323	Idler spindle felt washer.
	1314	Silent chain adjusting screw.		

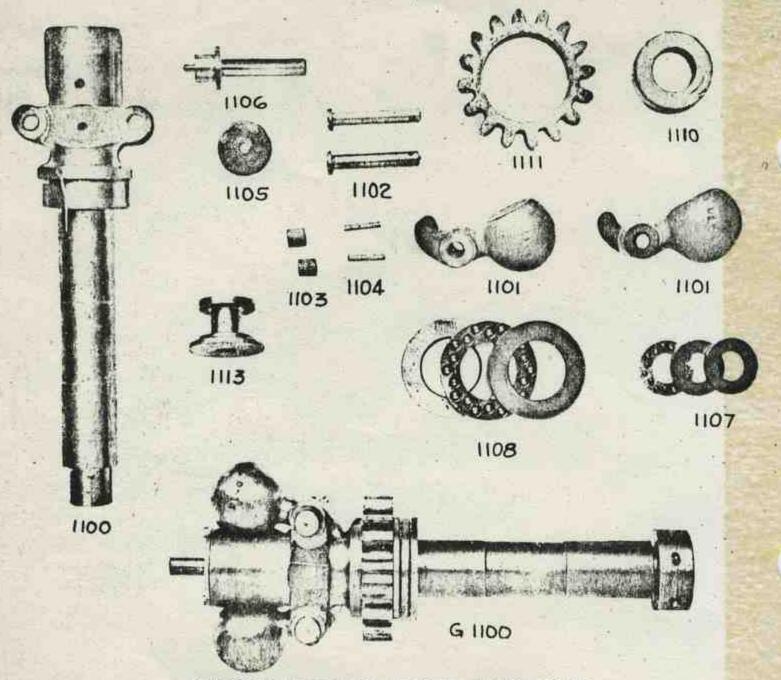
For Drawing showing Parts No. 1306, 1307, 1308, 1309, 1311, 1313, 1317, see page 29.



OIL COOLER

775 Oil cooler core. 777 Oil cooler head. 778 Oil cooler pipe.

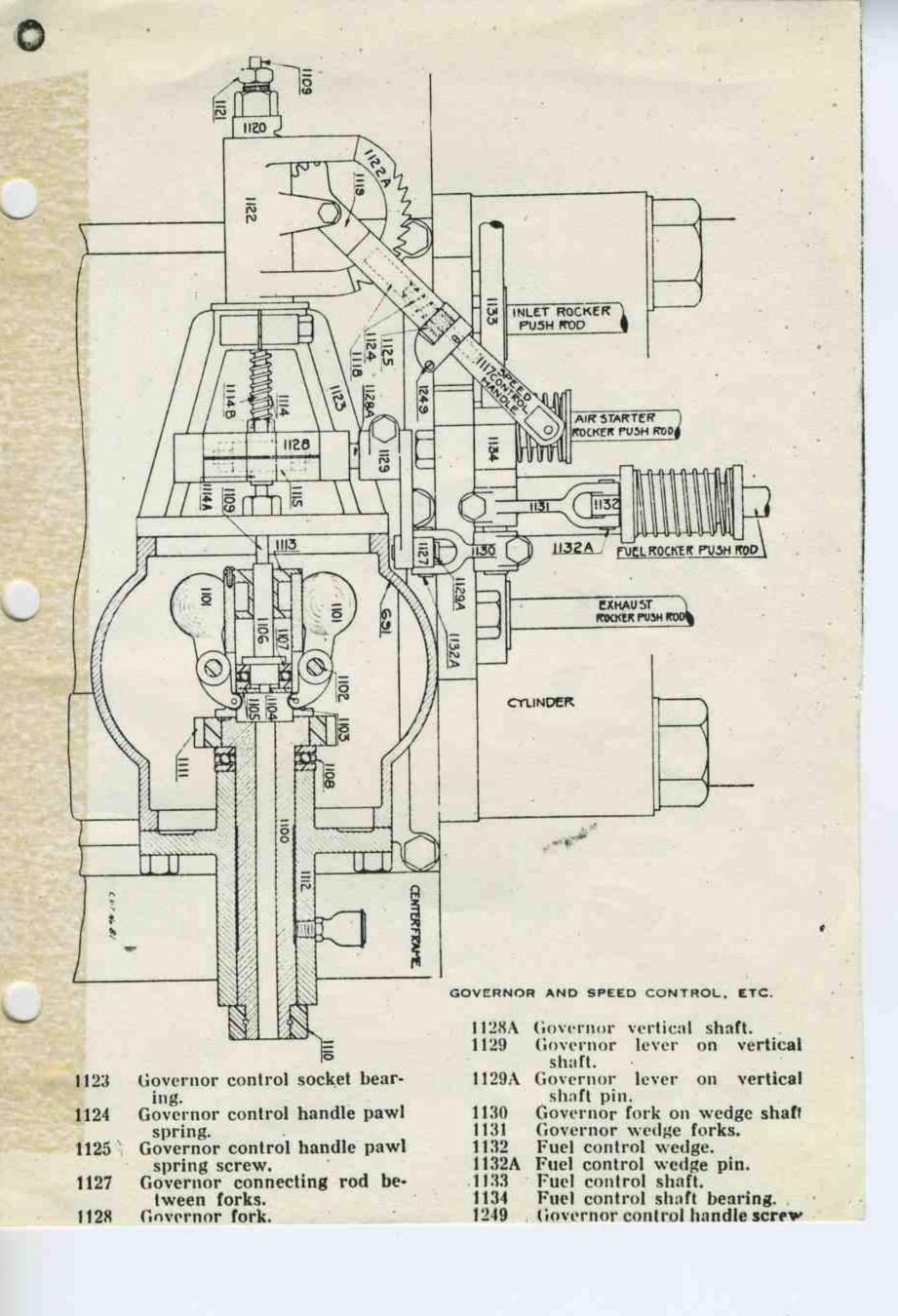
779 Oil cooler gland. 780 Oil cooler inlet pipe.

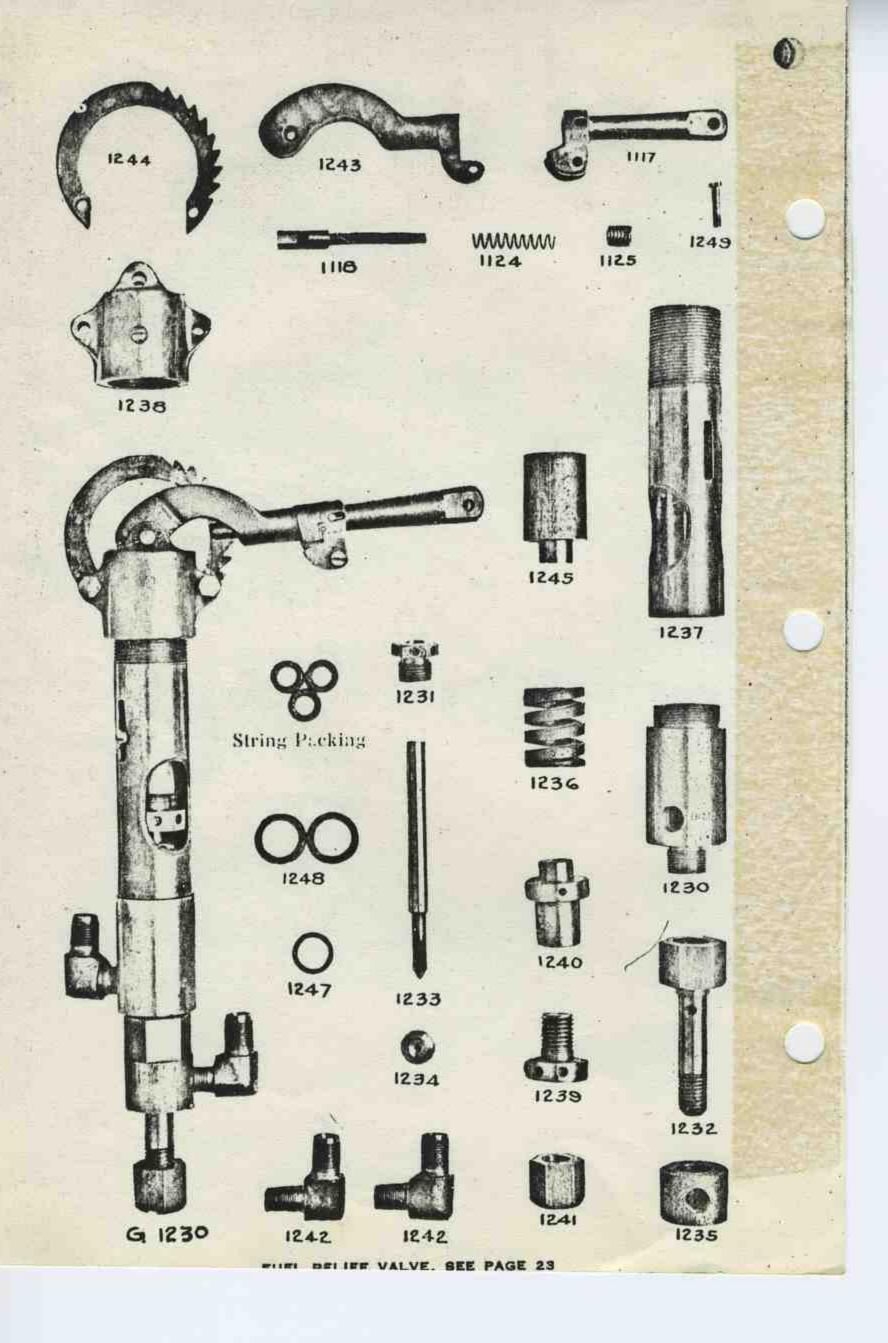


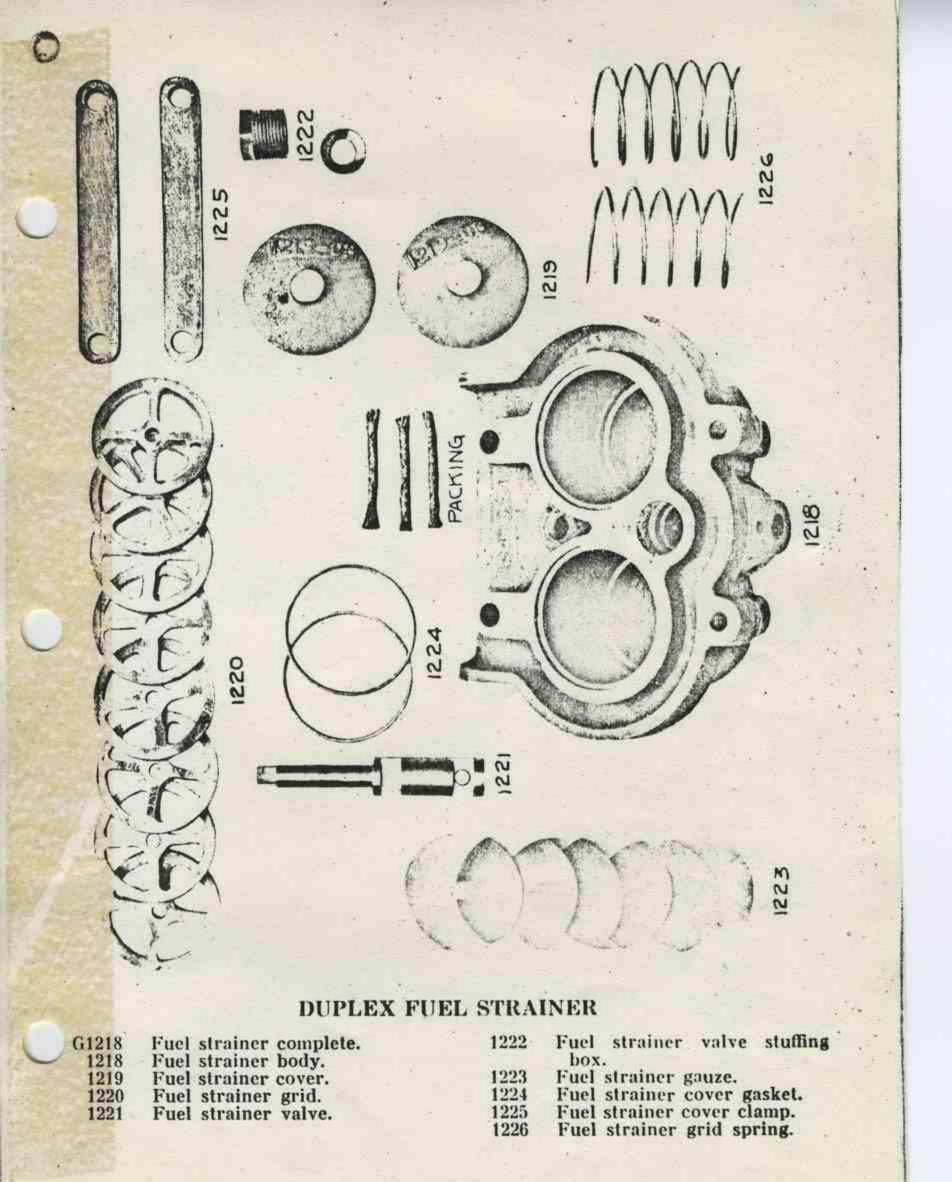
GOVERNOR AND SPEED CONTROL CONNECTIONS

Governor

			The second second
691	Governor shield.	1114A	Governor compression spring
691A	Governor shield gasket.		collar.
G1100	Governor assembly: Part Nos	1114B	Governor compression spring
37.4.4.10.10	1100, 1101, 1102, 1103, 1104,		collar.
	1105, 1106, 1107, 1108, 1109,	1115	Governor compression spring
	1110, 1111, 1113.		block.
GA1100	Governor body (always sup-	G1117	Governor control handle, com-
0111100	plied with 1111 fitted on).	100000000	plete: Part Nos. 1117, 1118
1101	Governor weight.		1119, 1124, 1125, 1249.
1102	Governor weight pin.	1117	Governor control handle.
1103	Governor weight roller.	1118	Governor control handle pawl
1104	Governor weight roller pin.	1119	Governor control handle sec-
1105	Governor weight roller plate.	1377778	tor.
1106	Governor thrust quill.	1120	Governor rack for compres-
1107	Governor thrust quill ball		sion spring.
1107	bearing, small.	1121	Governor rack adjusting screw
1108	Governor body ball bearing,	GA1122	
1100	large.		et with sector 1122A.
1109	Governor shaft.	1122A	
1110	Governor body collar.	112011	sector.
1111	Governor pinion.	G1122	Governor speed control socke!
1112	Governor bearing.	GIIZZ	and handle complete: Part
1113	Governor thrust quill bushing.		Nos. 1117, 1118, 1119, 1120,
1114	Governor compression spring.		1121, 1122, 1122A, 1124, 1125
1114	Governor compression spring.		1249.
WARDOW!			1670.







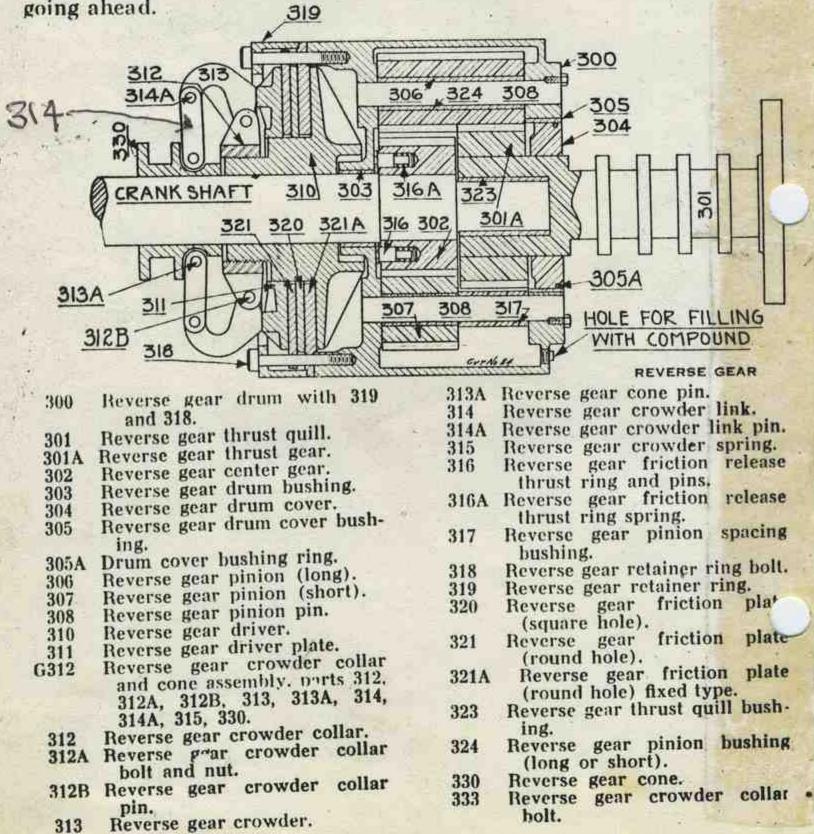
The gears, in the drum, when reversing should be constantly running in compound. Fill the drum with a good grade of compound (generally done at first in the factory) when needed, through the plugged hole in the after end of the drum. It is good practice to put about one quart of good transmission oil in drum every few weeks.

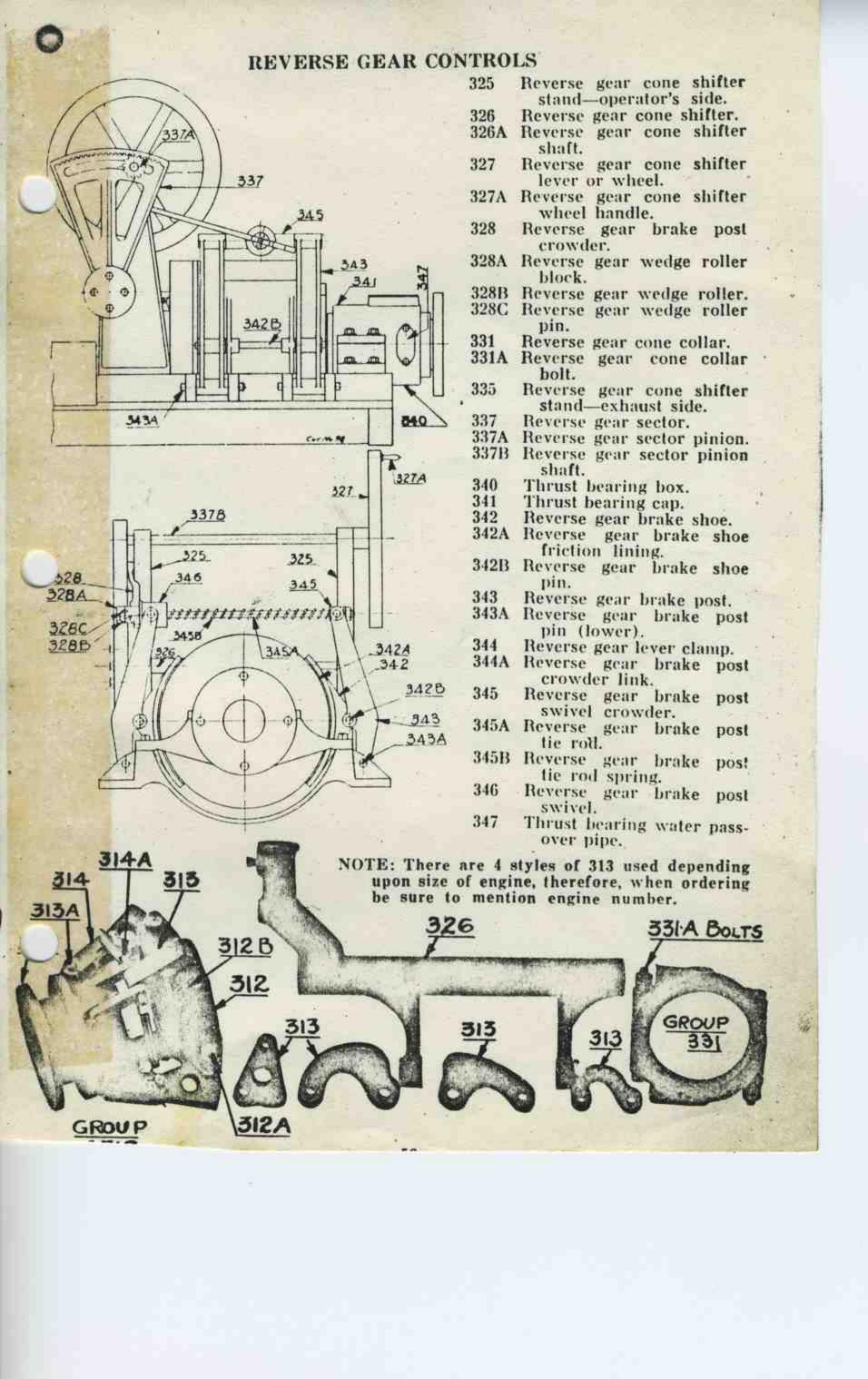
The friction clutch is of the multiple-disc type. In case the clutch begins to slip, loosen one of the bolts holding the crowder collar together and turn collar slightly in clockwise direction, thereby tightening on the threads of the driver. Be sure to tighten all bolts again before attempting to run engine.

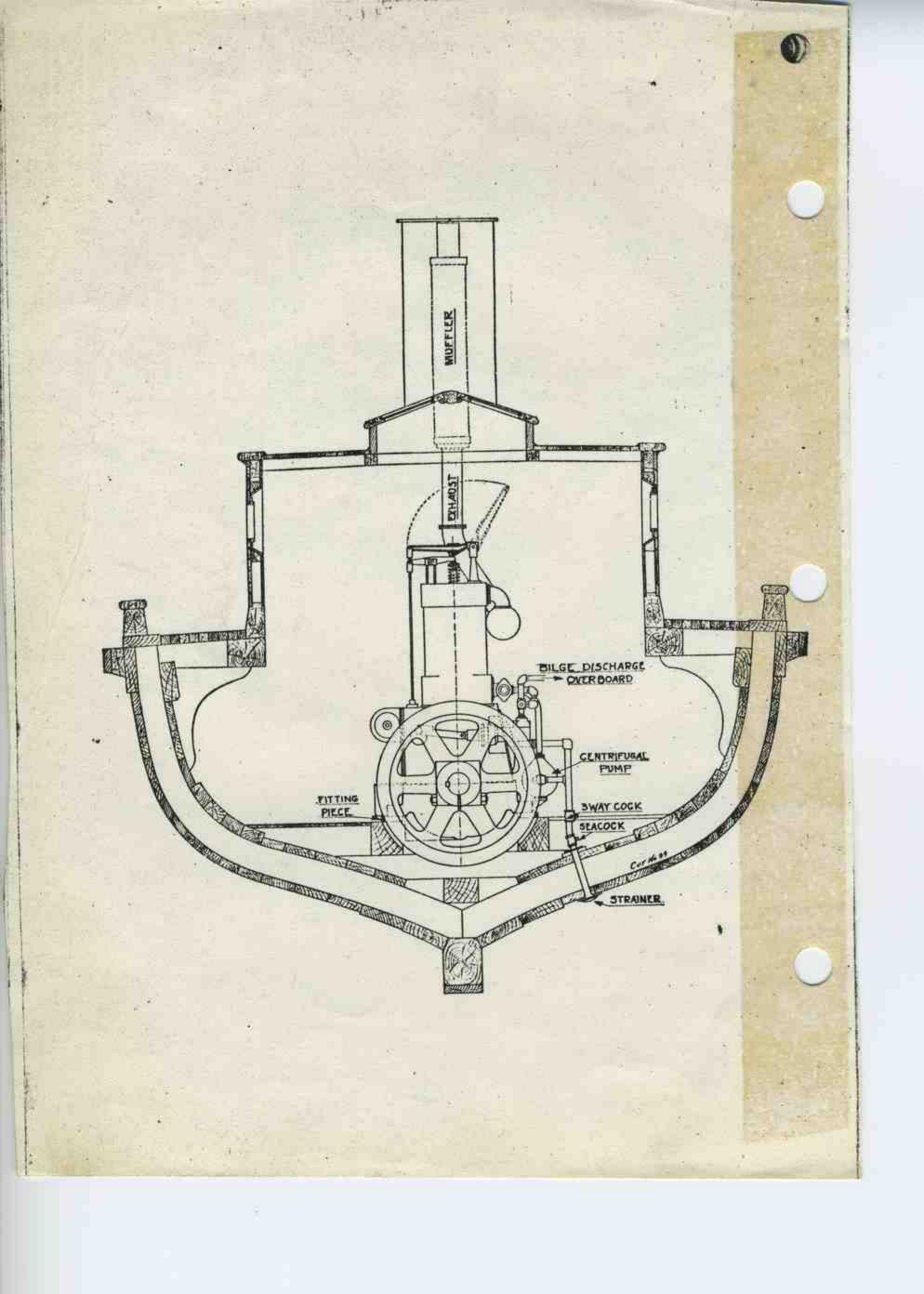
Plates should be oiled occasionally by pouring oil in the open end

of the reverse gear drum.

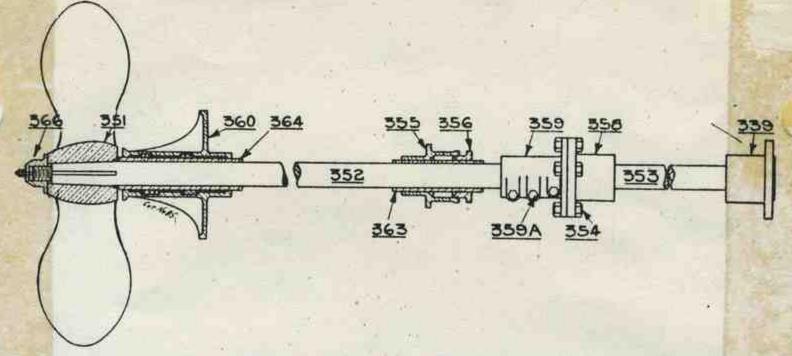
If the brake shoes, Part No. 342, become too loose, so as not to hold drum stationary when reversing, tighten tie rod connecting brake posts. Be sure that brakes shoes are entirely free of the drum when engine is going ahead.



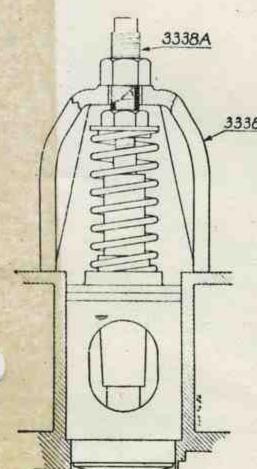




PROPELLER AND SHAFTING



		the state of the s		
	339 351	Reverse gear coupling. Propeller.	357 358	Stuffing box gland stud. Intermediate shaft coupling.
	G352	Propeller shaft complete with	359A	Propeller shaft coupling bolt.
		Parts 359, 359A, 363, 364, 366.	359	Propeller shaft coupling.
	G353	Intermediate shaft complete	360	Stern bearing.
		with Parts 339, 354, 358.	363	Propeller shaft bushing (stuff-
	354	Intermediate shaft coupling		ing box).
		bolt.	364	Propeller shaft bushing (stern
	355	Stuffing box.		bearing).
H	356	Stuffing box gland.	366	Propeller nut and lock screw.

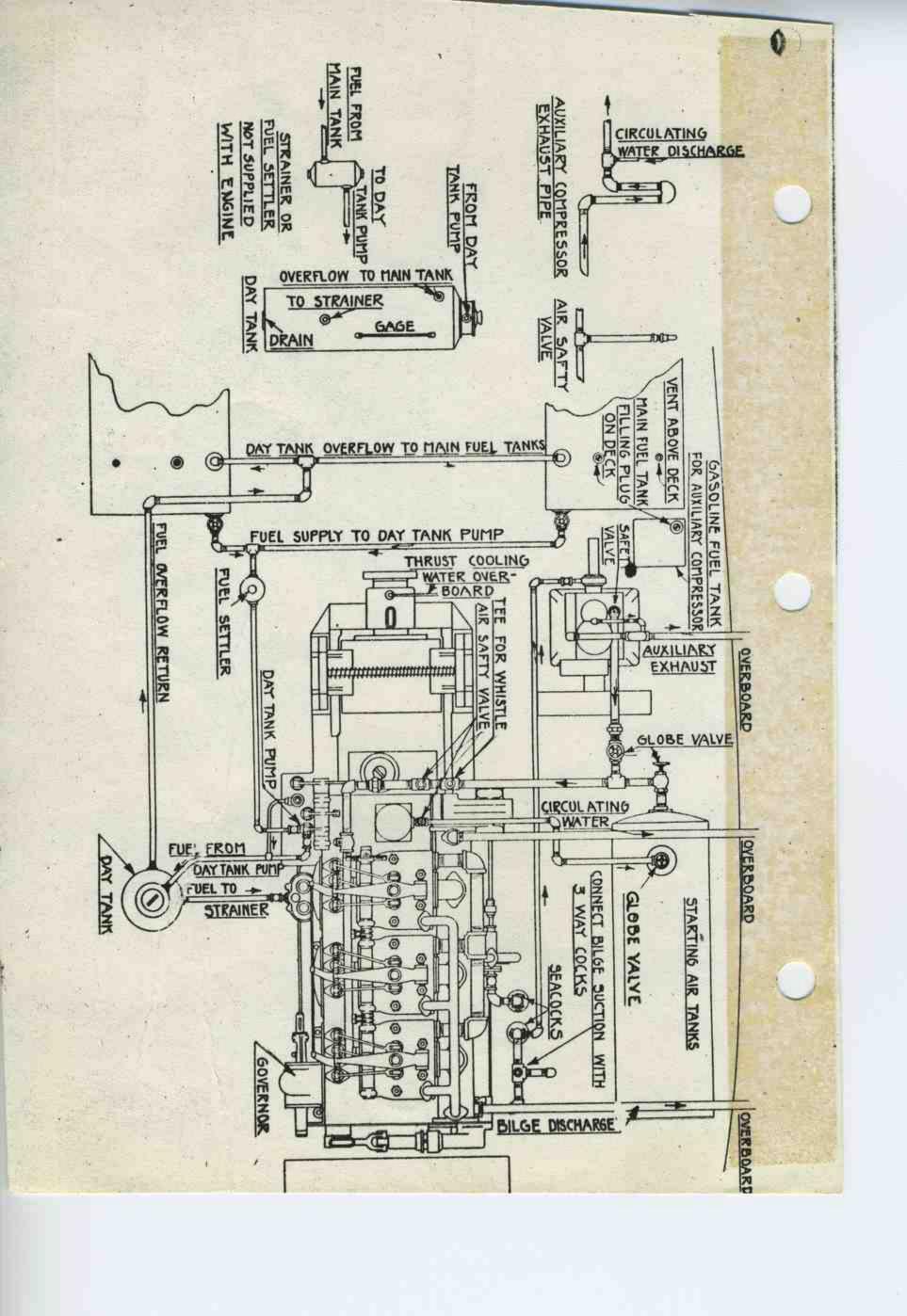


VALVE AND CAGE REMOVER

The cut opposite shows the proper way to remove valve in cages from the cylinder head. First remove the flange that holds the cage down, then remove the top lock nut of the valve.

Screw the stud over the valve stem all the way down until it is tight against the valve stem nut, then screw the large nut in a clockwise motion, which will raise the valve and cage enough to permit it to be readily removed from the cylinder head without any undue strain.

3338 Valve cage lifting yoke.
3338A Valve cage lifting yoke screw.



Marine Installation

THE condition of the boat, its length, breadth and the purpose for which it is intended to be used, will largely determine the location for the installation of the engine. Another point that must be considered in the selection of the engine location is that a suitable foundation must be arranged for. Just as the heavy duty gas engine requires a heavier hull construction and foundations than the light, high speed engine, so the diesel engine requires still heavier construction, because of its heavier power impulses, than the heavy duty gas engine.

It has been found from experience that engine timbers extending fore and aft as far as the lines of the boat will permit, are essential if satisfactory operation is to be obtained. We strongly recommend that these timbers be well fastened to the ribs and the keelson by means of through bolts and cross bolts. If the width of the engine base is such that in locating these foundation timbers a space is left between them and the keelson, this space should be filled in with solid timbers for at

least two or three feet fore and aft of the engine base.

Provision should be made in the form of long, tapered wedges equal in length to the length of the engine base, for alignment of the engine. In small engines, lag screws are used for fastening the engine to the foundation timbers, but in large engines the engine should be secured to the hull or foundation timbers by through bolts in at least

three points on each side of the engine base.

It is important that in lining the engine up the driving flange on the thrust shaft should be square with the flange on the propeller shafting. It is also desirable that the inclination of the engine to the water line be as small as possible. The hole through the deadwood is best made water-tight by means of a lead sleeve. This lead sleeve extends through the deadwood about 1" on either end. These extensions are flanged over to form a gasket under the stern bearing and stuffing box. The lead sleeve should be selected of such a diameter that it can be readily slipped through the hole in the deadwood, after which it is expanded tight against the deadwood by driving greased wooden plugs through it. After the lead sleeve has been expanded and flanged over, the stern bearing and stuffing box may be set in place after coating the flanges well with white lead and screwing up tight against the deadwood by means of lag screws furnished with the engine. After the propeller shaft and the engine are installed, it is important in determining the length of the intermediate shaft, that sufficient clearance be allowed between the hub of the propeller and the stern bearing to allow for a slight lengthening of the boat, due to the swelling after it is placed in the water. When installing the propeller, it is well to coat the tapered end of the propeller shaft and the threads with a good coating of white lead and machine oil. This will prevent salt water from getting in between and starting corrosion. Be sure that the propeller nut is put on tight.

ALIGNMENT

When installing the engine, use great care in alignment, which can be best done by first installing the tail shaft, then lining up and bolting the coupling of the intermediate shaft. Adjust the height and thwartship position of the engine so that the flanges of the engine and the intermediate shaft couplings are square with each other. The engine timbers should be set low enough to allow a soft, wooden shim about 1½" thick to be placed between the engine and engine timbers. After the engine is lined up, take measurements for the shims and cut them to fit snug. Check up the flanges of the engine and intermediate shafting couplings to make sure they are in perfect alignment before bolting up. Base is tapped for jacking up screws for use in lining up engine.

EXHAUST PIPING

The exhaust piping should be of a size to correspond with the threaded flange on the end of the water-cooled exhaust pipe on the engine. We recommend that the exhaust pipe be run straight up, using as few elbows as possible and as direct as convenient to the exhaust stack. Where elbows must be used in the exhaust pipe, use 45° elbows in place of 90° elbows. The muffler furnished with the engine should be attached to the exhaust pipe at a point to come within the exhaust stack. Standard black iron pipe and fittings are recommended for exhaust piping. The exhaust stack, while not furnished with the engine, may be constructed of about No. 16 gauge galvanized sheet iron with an angle iron flange about 2" x2" at the bottom for bolting to the deck of the boat. The top end of the stack should be re-inforced by heading over a ring or flanging. The exhaust stack dimensions should be such as to allow the muffler to pass readily up through the stack.

CIRCULATING WATER PIPING

All circulating water piping and fittings should be of brass of standard iron pipe sizes. The size of pipe to use is determined by the suction opening on the pumps. Locate and place the seacocks and flanges about 3 feet from the suction openings on the pump and run the pipes direct, with as few bends as possible, to the pumps. The sea fittings or flanges are installed on the ship's planking and securely fastened to same. The seacocks are to be attached to this fitting directly inside of the ship's planking and so located as to be readily accessible. The overboard discharge from the engine leads out from the forward end of the exhaust pipe and should be carried horizontally, if possible, to the side of the ship and there discharged overboard. The centrifugal pump is ordinarily used for circulating water, while the plunger pump is used as an auxiliary circulating pump and also for bilge pumping. The discharge from the plunger pump is connected up to the circulating system on the engine and a three-way cock is also provided which may be connected up to a separate overboard discharge for use when pumping the bilges and may also be used by connecting to a proper outlet for washing decks, fire or other general service. The suction from the plunger pump should be provided with a three-way cock, one opening of which is connected to a separate sea-cock and the other run down to the bilge and provided with a bilge strainer. It is important that all piping be kept out of the splash of the bilge water. We recommend that this pining he kent shove the engine room floor.

AIR RECEIVERS AND PIPING

The air receivers or air tanks should be installed as near as possible to the engine and connected to the starting air valve on the engine with a pipe of at least the size corresponding to the opening in the air tanks. Place the valve furnished with the engine directly on the air ank and provide an opening in this pipe for connecting up to the distance of the auxiliary air compressor. The starting air pipe should be connected in the opening on the end of the air tank. The air compressor on the engine is connected to the air tank by means of a separate pipe. Air piping should be black pipe, preferably of Buyers brand. USE NO GALVANIZED OR BRASS PIPING ON THE AIR LINE.

See suggested installation plan, page 53.

LUBRICATING OIL PIPING

On all, except the largest engines, the lubricating oil system and piping is attached to the engine proper and need not be interfered with in installing the engine. However, on the larger engines, where it is inconvenient to attach the lubricating oil service tank to the engine, it is furnished separate and intended to be supported on a bulkhead as close as possible to the engine. The discharge from the sump pump should be connected to the top of the lubricating oil service tank by rans of 1" Buyers black pipe. The side connection from the lubriting oil tank should be connected by means of 1½" brass pipe to the lubricating oil pressure pump. The discharge from the pressure pump is in all cases connected up to the lubricating system on the engine.

FUEL PIPING

The bottom of the main fuel tanks should be connected to the day tank pump with not less than ½ "Buyers black iron pipe. Provide a settler in an accessible place and as close as possible to the fuel tanks. The fuel piping should be kept above the floor line if possible to prevent coming in contact with the bilge water. The discharge from the day tank pump should be connected up to the top opening of the day tank with ½ "Buyers black iron pipe. The side opening of the day tank should be connected with the fuel oil strainer, using 1" brass pipe and reducing at the strainer. The overflow from the fuel day tank should be connected and run back to the main fuel tanks with ¾ "black iron pipe. In the placing of the fuel tanks as well as air tanks, it is 1 to remember to arrange the seams on the inside and otherwise

* 1

ke the tanks as accessible as possible for inspection or repair. It is also important to keep them out of the splash of the bilge water.

Auxiliary Engine

EXHAUST

The exhaust pipe from the auxiliary engine is usually carried overboard together with the discharge circulating water. It is usual to provide a gooseneck or return bend extending upward to a point higher than the water line, then returning down to a point still above the water line and allowing the discharge water from the top of the cylinder head to enter the exhaust pipe beyond the gooseneck. In this way the exhaust from the engine is silenced and is cooled so as not to burn the side of the ship where it passes through. The exhaust piping should be covered where it comes close to any woodwork to prevent scorching.

STATIONARY ENGINE INSTALLATIONS

The installation of stationary engines will vary so much with the service for which the engine is intended and the location, that it is hard to give more than a general description of the requirements.

FOUNDATION

The foundation in all cases should be sufficient to absorb the natural vibrations of the engine. The size of the foundations will depend entirely upon the soil conditions. Under ordinary circumstances, when the foundation is made of concrete, brick or masonry, it may be 5 to 8 ft. deep, depending upon the size of the engine, and if it extends a fe beyond the edge of the engine base, it is usually sufficient. However, the condition of the soil on which the engine is mounted will determine, in a large measure, the size of the foundation. In silty sediment it frequently happens that concrete or brick cannot be used and timber cribbing must be resorted to.

EXHAUST PIPING

Exhaust pipe can be of black iron pipe and may be run directly up and through the roof, using a muffler if desired. In many cases, it is desirable to run the exhaust pipe down under the floor and out through the side of the building. In any case, the exhaust pipe should be of the size of the flange furnished on the end of the exhaust pipe and should be as free as possible of bends, using 45° elbows wherever necessary. If the exhaust pipe is of any considerable length, it is well to increase the size.

CIRCULATING WATER PIPE

The circulating pump furnished on the engine will determine size of the suction pipe to be used. Connect up the pump suction we the source of supply with as short a pipe as possible and eliminate bends and elbows to the greatest extent. Where the cooling water supply is under pressure, it may be necessary to place a reducing valve in the line to prevent excessive pressures being placed upon the cylinder jackets. Pressure over 15 lbs. per sq. in. should be avoided. The discharge from the engine should be carried away in pipe of a size no smaller than the outlet water pipe on the engine. The cooling of the

engine may be effected by means of a circulating system, a cooling tower or a radiator. The circulating water piping may be wrought iron pipe, preferably galvanized, where a fresh water supply is available.

AIR PIPING

The air piping should be carried from the air tanks to the engine in as short a run as possible, avoiding elbows and short bends. The pipe size is determined by the opening in the air tanks and should be carried up directly to the air starting valve before reducing. Buyers black iron pipe is recommended for this service. Malleable iron fittings of extra heavy pattern should be used.

PREPARING THE ENGINE FOR STARTING

After the engine has been installed and all piping and connections have been properly made and tested, start the small air compressor and pump up the air starting tanks with a pressure of 200 lbs. per sq. in. When this pressure has been attained, test all of the safety valves and see that they are in proper working order. The time required to pump up the tanks varies with the size, but will average about 15 minutes.

Inspect the inside of the engine base to see that no tools have been left there to interfere with the operation of the engine. At the same time, it is well to see that all dirt, shavings, etc., left by workmen when installing, have been cleared out so as not to interfere with the proper fubrication of the engine. Pour sufficient lubricating oil into the engine base to submerge the suction valve on the sump pump. This requires about 2 or 3 inches of lubricating oil in the sump. Next fill the lubricating oil service tank about 2 3 full of lubricating oil and likewise fill up the sight feed pressure lubricator on the engine. See that all of the adjusting screws on the pressure lubricator are open so as to give an ample supply of lubricating oil to the cylinders. This can be tested by turning the hand crank on the pressure lubricator. After the engine has been run in for a while the adjustment on these lubricators can be attended to, giving each cylinder from 13 to 14 drops per minute. At this point it is well to go over the engine with an oil can, squirting a few drops into all oil holes on the rocker arms, fuel pump cranks, etc. Fill up the main fuel oil tanks with the proper grade of diesel fuel oil and put a small quantity into the fuel oil day tank.

Turn the engine over with the starting bar, being sure that all fuel isolating valves are closed and the cylinder snifter valves open. One turn of the engine in this manner will insure that everything is free and ear and that the engine can be safely turned over on air. Next pump ap by means of the hand priming pump, about 2000 lbs. on the oil pressure gauge. This pumping should be done when all of the fuel valves are closed. To be sure of this, place the engine with the pointer about 25° past top center on the flywheel. If any difficulty is experienced in pumping up pressure on the oil gauge, it probably indicates that there is air in the fuel lines. To relieve this air, loosen the union on the fuel rail which connects the pipe leading to the pressure gauge. It may be necessary also to relieve the air at the discharge union from the fuel

pumps. After the fuel pressure has been pumped up, it is well to check the opening and closing of the spray valve (see instructions under Spray Valve Timing). After the timing has been found in order, the engine is ready to turn over a few turns under air pressure with the snifter valves open. This will clean out any accumulation of fuel oil in the cylinders and prevent excessive pressures when starting.

Bar the engine over to a point about 25° past top center, making sure that both the inlet and exhaust valves are closed on the particular cylinder indicated. Close the snifter valves and open the fuel isolating valves. Set the fuel pressure relief valve handle three or four notches up from the lowest position and pump up about 2000 lbs. on the gauge. Set the governor speed control handle in about medium position and see that the starting handle is in the running position. (See Fig. A, page 20.) Now open the air starting valve on the air manifold and the engine is ready to start. By pulling the starting handle over into the starting position, the engine will turn over and immediately start under its own power. The handle should then be quickly returned to the running position so as to save air and the starting valve on the air manifold closed.

It is well to pump up the tanks to their full pressure after each starting of the engine. To do this it is not necessary to start the auxiliary engine air compressor as the air tanks can be pumped with the compressor on the main engine. After the tanks have been pumped up, as indicated by the popping of the safety valves, the shut-off valves on the tanks themselves should be closed and the air compressor cut out by holding down the suction valve by means of the handle provided on the compressor cylinder head.

In pumping up the air tanks it is well to allow the safety valves to pop each time as this is a good check on the proper functioning of the valves.

Once a day, or each time the engine is started, it is well to squirt a mixture of cylinder oil and kerosene on the valve stems to prevent them from sticking.

PARTS LIST

For Atlas Imperial Diesel Engines

INSTRUCTIONS

In ordering parts BE SURE to give the following information: First—Name of Part Wanted.

Second-Number of Part Wanted.

Third-ENGINE NUMBER (found on name plate on centerframe).

IMPORTANT-RECHECK ALL NAMES AND NUMBERS TO BE SURE THAT THE CORRECT INFORMATION IS GIVEN.

CI	RCULATING WATER	Part No	D. Part Name
		305A	
Part No		306	Reverse gear pinion (long).
214		307	Reverse gear pinion (short).
214A	Plunger pump valve cage stud	308	Reverse gear pinion pin.
215	Plunger pump suction bonnet.	310	Reverse gear driver.
215A	Plunger pump bonnet copper	311	Reverse gear driver plate.
	asbestos gasket.	G312	Reverse gear crowder collar
215B	Plunger pump suction bonnet		and cone assembly, Parts
De la Company	flange.		312, 312A, 312B, 313, 313A,
216	Plunger pump discharge bon-		314, 314A, 315, 330.
	net.	. 312	Reverse gear crowder collar.
216A		312A	Reverse gear crowder collar
1	net stud.	0.2.1	bolt and nut.
217	Plunger pump valve guide	312B	Reverse gear crowder collar
Carrier W	stem.	O'LLIS	pin.
218	Plunger pump valve spring.	313	Reverse gear crowder.
219	Plunger pump valve (rubber)		Reverse gear cone pin.
219A	Plunger pump valve bushing.	314	Reverse gear crowder link.
G260	Plunger pump complete: Parts		Reverse gear crowder link
	219A, 260, 261, 262, 262A,		pin.
C+000	262B.	315	Reverse gear crowder spring.
GA260	Plunger pump body with	316	Reverse gear friction release
nn.	262A.		thrust ring and pins.
261	Plunger pump plunger.	316A	Reverse gear friction release
262	Plunger pump gland.		thrust ring spring.
262A	Plunger pump gland stud.	317	Reverse gear pinion spacing
262B G264	Plunger pump gland stud nuts		bushing.
0204	Plunger pump connecting rod	318	Reverse gear retainer ring
2644	with 264A.		bolt.
204A	Plunger pump connecting rod	319	Reverse gear retainer ring.
9640	bushing.	320	Reverse gear friction plate
2040	Plunger pump connecting rod		(square hole).
266		321	Reverse gear friction plate
266A	Plunger pump crank disc.	2210	(round hole).
10000	Plunger pump crank disc washer.	321A	Reverse gear friction plate
3 3	madici.		(round hole) fixed type.
300	Reverse gear drum with 319	323	neverse gear thrust quill
	and 318.	004	bushing.
301	Reverse gear thrust quill.	324	Reverse gear pinion bushing
100 00000	Reverse gear thrust gear.	907	(long or short).
The second second second	Reverse gear center gear.	325	Reverse gear cone shifter
	Reverse gear drum bushing.	200	stand—operator's side.
	Reverse gear drum cover.	326	Reverse gear cone shifter.
	Reverse gear drum cover	326A	Reverse gear cone shifter
-	bushing.		shaft.

Part No.	Part Name	Part No.	Part Name
327	Reverse gear cone shifter lev-	G370	Centrifugal pump complete:
	er or wheel.		Parts 370, 371, 371A, 372, 373,
327A	Reverse gear cone shifter wheel handle.		373A, 374, 379, 381, 381A, 382, 383.
328	Reverse gear brake post	GA370	Centrifugal pump body with
	crowder.		374.
328A	Reverse gear wedge roller	370A 371	Centrifugal pump outlet flang
328B	block. Reverse gear wedge roller.	371A	Centrifugal pump cover. Centrifugal pump cover screw
328C	Reverse gear wedge roller pin	371B	Centrifugal pump inlet flange.
330	Reverse gear cone.	G372	Centrifugal pump runner and
331	Reverse gear cone collar.	979	shaft.
331A	Reverse gear cone collar bolt.	373 373A	Centrifugal pump gland. Centrifugal pump gland stud.
333	Reverse gear crowder collar bolt.	374	Centrifugal pump shaft bush-
335	Reverse gear cone shifter		ing.
	stand—exhaust side.	379	Centrifugal pump shaft.
337	Reverse gear sector.	381	Centrifugal pump steady bear- ing.
337A 337B	Reverse gear sector pinion. Reverse gear sector pinion	381A	Centrifugal pump steady bear-
00713	shaft.	********	ing cap screw.
339	Reverse gear coupling.	382	Centrifugal pump shaft collar
339A	Reverse gear coupling bolt.	383 384	Centrifugal pump ball bearing Centrifugal pump shaft coup-
340 341	Thrust bearing box. Thrust bearing cap.	904	ling (complete).
342	Reverse gear brake shoe.	G500	Cylinder head complete as-
342A	Reverse gear brake shoe fric-		sembly (as shown on page
0.400	tion lining.		12), except spray valve G849, and horse shoe collar 877,
342B 343	Reverse gear brake shoe pin. Reverse gear brake post.		the following parts are
343A	Reverse gear brake post pin		cluded, Nos. 500, 510,
200	(lower).		513, 514, 514A, G520, G527A,
344			G550, 564, 564B, 565, 566, 567, 567A, 567B, 579, 580, 582, 583,
344A	Reverse gear brake post crowder link.		584, 585, 586, 588, G590, 764,
345			854, 854A, 855, 855A, G870,
	vel crowder.		872, 873. Specify whether handle or
345A	Reverse gear brake post tie rod.		male connecting link is
345B	Reverse gear brake post tie		wanted with head. The fol-
	rod spring.		lowing parts are also in-
346	Reverse gear brake post swi-		cluded if used: 515, 516, 517.
247	vel.	GA500	Cylinder head with exhaust inlet, air starting valves
347	Thrust hearing water pass- over pipe.		and springs only. Part Nos.
	NUMBER OF STREET		500, 510, 512, 513, 514, 514A,
PRO	PELLER EQUIPMENT		564, 579, 580, 582, 583, 584,
351	Propeller.	500	585, 586, 588. Cylinder head only with studs
G352	Propeller shaft complete with Parts 359, 359A, 363, 364, 366.	500	567 and 764.
G353	Intermediate shaft complete	501	Cylinder head stud.
	with 339, 354, 353.	501A	
354	Intermediate shaft coupling	502	for lifting head. Cylinder head gasket.
355	bolt. Stuffing box.	510	Valve and stem, inlet or ex-
356	Stuffing box gland.	CATALON .	haust.
357	Stuffing box gland stud.	512	Valve stem bushing, inlet or
358	Intermediate shaft coupling.	513	exhaust. Valve spring, inlet or exhaust
359 359A	Propeller shaft coupling. Propeller shaft coupling bolts.	514	Valve spring bushing, inlet or
360	Stern bearing.		exhaust—top.
363	Propeller shaft bushing (stuff-	514A	Valve spring bushing, inlet or exhaust—bottom.
201	ing box).	G515	Inlet or exhaust valve and
364	Propeller shaft bushing (stern bearings).	Cioro	cage complete: Parts 510
	****		819 StA and 515.

Part Name Part No. Part Name 578 Valve cage, inlet or exhaust Air starting pipe shut off valve 515 with 512 in engines 11" bore (state whether globe, gate or quick opening valve). and larger. 579 Air starting valve spring 516 Valve cage stud. bushing. Valve cage flange. 517 G580 Air starting valve complete. Valve cage gasket. 518 Cage type—Parts No. 580, 581, 519 Exhaust or inlet valve cage 581A, 582, 582A, 584, 585, 585A, 586. (See page 19 for ring. drawing.) G520 Inlet rocker, with roller and 580 Air starting valve. Parts 520, 520A, 523, Air starting valve cage. 581 524, G527A. 581A Air starting valve cage ring. Inlet rocker only. 520 581B Air starting valve cage gas-520A Inlet rocker bushings (set of 2) ket. 523 Rocker roller, for inlet, ex-582 Air starting valve spring. haust or air. 582A Air starting valve spring nut. 524 Rocker roller pin, for inlet. 583 Air starting valve spring washer—top. exhaust or air. 584 Air starting valve nut. Inlet or exhaust valve push 525 585, Air starting valve balance rod. bushing. 526 Push rod fork, for inlet, ex-585A Air starting valve balance haust, air or fuel. bushing nut. Fork pin, inlet, exhaust, air 527586 Air starting valve balance or fuel. bushing ring. G527A 588 Air starting valve spring Fork pin, ball lock. Inlet or exhaust valve lifter G528 washer-bottom. Air starting rocker with roller with roller and pin. G590 529 Exhaust or inlet valve lifter and pins. 523, 524, G527A, guide. (and 590A when used). 530 Exhaust or inlet valve lifter Air starting valve rocker roller. roller. Part 523. 531 Exhaust or inlet valve lifter Air starting valve rocker roller pin. roller pin. Part 524. 532 Inlet and air starting cam. 590A Air starting valve rocker Exhaust rocker with roller G550 bushing (set of 2). and pins, 550, 550A, 523, 524, Air starting valve rocker fork. G527A. Part 526. 550A Exhaust rocker bushings (set Air starting valve rocker fork of 2). pin. Part 527. 560 Exhaust cam. Air starting valve rocker fork 564 Rocker shaft, connecting link pin lock. Part 527A. (female). 593 Air starting valve push rod. 564A Rocker shaft connecting link G594 Air starting valve lifter with (male). roller and pin. 564B Rocker shaft connecting link 594A Air starting valve lifter collar pin. 594B Air starting valve lifter spring 565 Rocker shaft. 595 Air starting valve lifter rol-Rocker shaft bearing (state: . 566 ler. with or without wings). 596Air starting valve lifter rol-567 Rocker shaft bearing stud. ler pin. 567A Rocker shaft bearing stud 599Air starting valve lifter guide. nut; top. 600 Cylinder. 567B Rocker shaft bearing stud Cylinder clean out cover. 605 lock nut. 610 Cylinder water by-pass (cast G570 Air starting handle, complete: iron). Part Nos. 570, 571, 571A, Cylinder head water pipe 610 572, 573. (brass tube). 570 Air starting handle. Cylinder head water pipe 610A 571 Air starting handle pawl. rubber gromet. 571A Air starting handle pawl screw 611 Cylinder lubricating pipe. Air starting handle pawl 572 612 Cylinder lubricating pipe gland spring. Cylinder lubricating pipe 573 Air starting handle sector. packing ring. 577 Air starting pipe elbow.

Cylinder

elbow.

lubricating

pipe

Part Name Part No. Part Name Part No. 722A Base cap shim-after end. 620 Piston. Base to cylinder stud. 727Piston pin. G621 Base to centerframe stud 729 622 Piston ring. (cylinder). Connecting rod bushing (pis-628 Cylinder to centerframe stud. 730 ton pin). Base to centerframe stud (air 731 Connecting rod. 630 Connecting rod check valve. pump). G632750 Flywheel with 753. Connecting rod foot shims 634 Flywheel hub clamp bolt. 753 (state thickness). Flywheel hub clamp bolf Crank pin bearing (2 halves) 754 636 wrench. babbitted. Exhaust elbow (on cylinder Crank pin bearing shims 760 636A head). (state thickness). Exhaust pipe. 761 Crank pin bearing bolts and 637 Exhaust pipe blind flange. 763 Exhaust elbow stud (on cyl-Crank pin bearing bolts lock-764 638 inder head). ing strap. Exhaust elbow (on end ex-765 Crankshaft with 660 and 655. 640 haust pipe). Main bearing shells-flywheel 644 Stud for exhaust elbow (on 766 end (2 halves). end exhaust pipe). Main bearing shells-center 646 Exhaust flange for elbow (on 767 (2 halves). end exhaust pipe). Main bearing shells-after 649 Lubricating oil cooler core. 775 end (2 halves). Lubricating oil cooler head. 777 Crankshaft oil throw ring. 655 Lubricating oil cooler pipe. 778 Crankshaft pinion. 660 Lubricating oil cooler gland. 779 Intermediate gear (including 664 Inlet water manifold (or oil 780 band 664A). cooler body). Intermediate gear-pin. 665 Inlet water manifold end 784 665A Intermediate gear-pin bolt. flange (threaded). State pir 665B Intermediate gear-pin bolt size. washer. Inlet water manifold end 665C Intermediate gear-pin washer 786 flange (blind). Inte mediate gear bearing. Inlet water manifold side 666A Intermediate gear bearing cap 787 flange (state pipe size). screw. Outlet water manifold. 789 666G Intermediate gear bearing Outlet water pipe-outlet 791 lock set screw. from manifold. Cam shaft gear. 670 Outlet water pipe flange. 791A Cam shaft. 672 Water passover pipe on ex-792 Cam shaft bearing and bush-G680 haust pipe. ing. Outlet water pipe-manifold 794 Centerframe blind flange-684 to exhaust pipe. end of cam shaft. Outlet water pipe gland for 795 Cam shaft bushing. 685 794. Centerframe. 690 High pressure fuel discharge 796 Centerframe cover and gov-691 fitting. ernor shield. 796A High pressure fuel dicharge Governor shield gasket. 691A fitting split ring. Centerframe cover-round. 692 High pressure discharge fit-797 692A Round centerframe cover ting nut. gasket. H. P. Fuel pump complete: G800 Centerframe cover-air pump 693 Parts 800, 805, 806, 808, 800 Centerframe cover-end of 810, 810B. 695 High pressure fuel pump bo. 800 frame. High pressure fuel pump suc-801 Base with caps and studs. 710 tion valve. Base cap-flywheel end. 712 H. P. Fuel pump suction GE02Base cap-center. 713 valve cage with 801 and 811. Base cap-after end. 715 High pressure fuel pump suc-802 Base cap studs. 717 tion valve cage. Base cap parting piece-fly-719 High pressure fuel pump suc-802A wheel end. tion union nut. 719A Base cap shim-flywheel end. High pressure fuel pump suc-Base cap parting piece-center tion union sleeve. 720A Base cap shim-center. High pressure fuel pump --- con parting piece-after

rt No. Part Name Part No. Part Name 805A High pressure fuel pump 850 Spray valve stem. plunger spring. 851 Spray valve body. 806 High pressure fuel pump 853 Spray valve spring casing. plunger packing rings. 854 Spray valve clamp. 808 High pressure fuel pump gland 854A Spray valve clamp stud. 809 High pressure fuel pump gland 855 Spray valve clamp bridge. Spray valve clamp stud nut. nut. 855A 810 High pressure fuel pump 856 Spray valve seat nut. plunger head. 857 Spray valve spring plug. 810B High pressure fuel pump 858 Spray valve spring. plunger nut. 859 Spray valve ball bearing. 811 High pressure fuel pump dis-860 Spray valve gasket. 864 charge valve. Spray valve nozzle or tip. 817 High pressure fuel pump plate 865 Spray valve gland nut. 818 High pressure fuel pump 866 Spray valve gland. rocker (double). 867 Spray valve packing seat. 818A High pressure fuel pump 870 Spray valve rocker. rocker pin (long). Spray valve rocker pin No. 527 818B High pressure fuel pump Spray valve rocker fork No. rocker (single). 526. 818D High pressure fuel pump Spray valve rocker fork pin rocker bushing (bronze). lock No. 527A. 819 High pressure fuel pump 872 Spray valve rocker stand. rocker shaft. 873 Spray valve rocker stand pin. 819A High pressure fuel pump 876 Spray valve push rod. rocker shaft sleeve (steel). Spray valve horseshoe collar. 877 High pressure fuel pump G820 878 Spray valve stem nut. rocker shaft bearing and Spray valve stem lock nut. 878A caps. 880 Spray valve cam toe. 820A High pressure fuel pump 881 Spray valve cam disc. rocker shaft bearing stud G883 Spray valve lifter with roller (long). and pin. 820B High pressure fuel pump 884 Spray valve lifter roller. Spray valve lifter roller pin. rocker shaft bearing stud 885 (short). 887 Spray valve lifter guide. 820D High pressure fuel pump 888 Spray valve lifter spring. rocker shaft bearing cap 889 Spray valve purh rod socket. stud. AIR COMPRESSOR (on Engine) 821 High pressure fuel pump rocker shaft bearing cap. Air compressor cylinder 822 High pressure fuel pump with 902. crank · shaft. G901 Air compressor head, com-822A High pressure fuel pump plete: Parts 901, 905, 907, 908, crank shaft coupling. 908A, 915, 915A, 918, 918A, 822B High pressure fuel pump 919, 920, 922, 922A, 923, 924. crank shaft coupling bolts. 901 Air compressor cylinder head 822C High pressure fuel pump with 907. crank shaft washer. 902 Air compressor cylinder head G823 H. P. Fuel pump crank shaft stud. connecting rod assembly: 905 Air compressor inlet valve. Parts 823, 824, 824A. Air compressor valve grid. 906 824A High pressure fuel pump Air compressor inlet valve 907 crank shaft connecting rod bushing. cap bolts. Air compressor inlet valve 908 828 High pressure fuel pump conspring. necting link. 908A Air compressor inlet valve 828A High pressure fuel pump connecting link pin. 908B Air compressor inlet valve 829 High pressure fuel pump washer. crosshead guide. 620 Piston. 831 High pressure fuel pump G621Piston pin. crosshead. 622 Piston ring. Specify if plain, 831A High pressure fuel pump double seal or oil control, crosshead nut. Also specify width. 831B High pressure fuel pump Connecting rod bushing (pis-628 crosshead button. ton pin). G849 Spray valve complete. 630 Connecting rod.

10 20 4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		n	Dant Hama	
Part No.			Part Ilame	- 10
915	Air compressor discharge	1076A	Lubricating sump pum	con-
	valve.	1070	necting rod pin. Lubricating sump pump	crank
915A	Air compressor discharge	1079	disc.	
	valve adjusting screw.	10704	Lubricating sump pump	crank
918	Air compressor discharge	1019A	dise washer.	
	valve nut.	G1080	Pressure lubricating	pump
	Air compressor discharge	01000	complete: Parts 1080	
agata, in	valve relief handle nut.		1082, 1083, 1085.	
918B	Air compressor inlet valve	1080		pump
0400	cap nut stud. Air compressor inlet valve	2000	cylinder.	
918C	Genge stud	1080A	Pressure lubricating	pump
Oton	flange stud. Air compressor inlet flange		suction check valve.	- transmission - "
9101	gasket.	1080B	Pressure lubricating	pump
919	Air compressor discharge	- Townson	discharge check valv	e.
310	valve spring.	1081	Pressure lubricating	pump
920	Air compressor discharge		plunger.	pump
40.11	valve guide.	1081A	Pressure lubricating	pump
922	Air compressor discharge	4000	plunger nut. Pressure lubricating	pump
11	valve can nut.	1082		Linib
922A	Air compressor discharge re-	1083	gland. Pressure lubricating	pump
400	lief handle.	1000	gland nut.	
923	Air compressor discharge	1085	Pressure lubricating	pump
- 14	valve guide cap.	1000	plunger eye.	
924	Air compressor discharge	1085A		pump
4	valve spring bushing.		plunger eye pin.	Total Control Control
925	Air compressor piston. Air compressor piston ring.	1086	Pressure lubricating	pump
926	Air compressor piston pin.		connecting link.	minin
927	Air compressor piston pin	1087	Pressure lubricating	pump 0 lbe)
928	Lesching	n es	oil pressure gauge (3	pun
- UG929	Air compressor connecting	1088	Pressure lubricating	Punt
0020	nod hushed	40004	rocker. Pressure lubricating	pump
G930	Air compressor eccentric	10008	rocker nin.	
Trick	stean (9 halves) and Dolls.	G1090	Drossure Inbricating	relief
932	Air compressor eccentric strap	GIOOO	valve complete: Par	IS 1000,
Mouth	bolts.		1001 1092 1093 1095	. 1090.
933	Air compressor eccentric. Air compressor relief valve.	1090	Lubricating oil pressu	re rener
1811 1935	Air compressor pass over pipe		nolvo body can.	
936	Air compressor gauge (300	1091	Lubricating oil pressu	He remer
121 1937	nounds)		valve plunger. Lubricating oil pressu	re relief
1060	Cight feed pressure lubricator.	1092	valve adjusting scre	W.
1000	When ordering lubilcator	4000	Lubricating oil pressu	re relief
PLANTED VIEW	state manufacturer, number	1093	valva chring.	
- weight 3th 1	of feeds also type.	1095	Imbrigating oil pressu	ire relief
1061	Sight feed pressure lubricator	1000	valve adjusting scre	W Hut.
A Selveril	bracket.	1096	Lubricating oil pressi	He rener
1065	Sight feed pressure lubricator	V V EVI	chack valve and De	uv.
in the state	A Lubricator drive rod pin.	G1100	Covernor assembly	Parts
1065	Lubricating sump pump gland	5.6	1100, 1101, 1102, 11	08 1109.
1066			1105, 1106, 1107, 11	00, 1100,
1066	etud.	4400	1110, 1111, 1113. Governor body with	n pini
1069	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	1100	No. 1111.	
1003	strainer.	1101		
G107	o Sump nump assembly.	1101 1102	Governor weight pin	
4070	I uhricating sumb bump body.	1103	Governor weight ro	Her.
1073	A Lubricating sump pump check	1104	Covernor weight rol	er pm.
	valve_intet.	1105	Governor weight rol	ler plate.
1073	BB Lubricating sump pump check	1100	Covernor thrust dul	11.
	valve—outlet.	1107	Governor thrust q	uni ban
107	Lubricating sump pamp plan		bearing (small).	bearing
107	ger. Lubricating sump pump con-	1108	Governor body bal	Dearing
107	Lubricating sump pump con-		(large).	

Part Name togg Part No. Part Name art No. Governor shaft. 1109 HIGH PRESSURE FUEL PIPES Governor body collar. 1110 Each pipe assembled with nuts and Governor pinion. 1111 sleeves, Parts Nos. 1176, 1177. Governor bearing. 1112 Governor thrust quill bushing Testing pipe-fuel rail-to 1113 spray valve test clamp. 1114 Governor compression spring. 1184 Fuel pipe from relief valve 1114A Governor compression spring discharge-to pump suction. collar. 1185 Fuel pipe from high pressure 1114B Governor compression spring fuel pump discharge-to fuel Governor compression spring relief valve. 1115 1186 Connecting pipe between H. block. P. fuel pumps (discharge). G1117 Governor control handle, Fuel pipe from fuel pump-to complete: Parts 1117, 1118, 1187 1119, 1124, 1125, 1249. fuel rail. 1188 Fuel pipe from fuel rail-to 1117 Governor control handle. Fuel relief valve handle. spray valve. 1117 1189 Fuel pipe from fuel rail-to 1118 Governor control handle pawl Fuel relief valve handle pawl. 1118 fuel oil receiver. 1119 1189A Fuel pipe from fuel oil re-Governor control handle secceiver - to fuel pressure tor. 1120 Governor spring rack. gauge. Cylinder relief valve com-G1190 1121 Governor spring rack adjustplete: Parts 1190, 1191, 1192, ing screw. G1122 Governor speed control socket 1193, 1194, 1195, 1199, and handle complete: Parts 1190 Cylinder relief valve body. 1117, 1118, 1119, 1120, 1121, 1191 Cylinder relief valve stem. 1122, 1122A, 1124, 1125, 1249. 1192 Cylinder relief valve seat. GA1122 1193 Governor speed control sock-Cylinder relief valve spring. 1194 et with sector, 1122A. Cylinder relief valve spring 1122A Governor speed control socket collar. 1195 Cylinder relief valve adjustsector. 1123 Governor control socket bearing screw lock. 1196 Cylinder relief valve plug. Fuel relief valve handle pawl Snifter valve complete: Parts 1124 G1197 1197, 1198. Governor speed control pawl 1124 1197 Cylinder snifter valve body. 1198 Cylinder snifter valve stem. spring. 1125 1199 Fuel relief valve handle pawl Cylinder relief valve adjustspring screw. ing screw. 1125 Fuel rail complete with iso-1200 Governor speed control pawl spring screw. lating valves. 1127 Governor connecting rod be-1201 Spray valve fuel strainer, body tween forks. 1202 Spray valve fuel strainer stem 1128 Governor fork. 1203 Fuel rail clamps. 1128A Governor vertical shaft. 1204 Spray valve fuel strainer gas-1129 Vertical shaft lever. ket. 1129A Vertical shaft lever pin. 1205 Fuel rail isolating valve body. 1130 Wedge shaft fork. 1206 Fuel rail isolating valve stem. 1131 Wedge fork. 1206 Fuel oil receiver valve stem. *When combined in one cast-1207 Fuel rail isolating valve gland ing order Part No. 1130A. -1207Fuel oil receiver valve gland. Fuel control wedge. 1132 1208 Fuel rail isolating valve gland 1132A Fuel control wedge pin. nut. 1133 Fuel control shaft. 1208 Fuel oil receiver valve nut. 1134 Fuel control shaft bearing. 1214 Fuel oil receiver. 1135 Fuel control shaft spring. G1215 Fuel oil receiver valve com-1136 Fuel control shaft spring clamp plete. 1139 Guard rail. 1215 Fuel oil receiver valve body. 1140 Guard rail bolt. 1216 High pressure fuel oil gauge 1141 Guard rail spacer. bracket. 1143 Fuel relief valve handle. 1217 High pressure fuel oil gauge. 1149 Fuel relief valve handle screw G1218 Fuel oil strainer complete. 1170 High pressure union nipple. 1218 Fuel oil strainer body. Fuel oil strainer cover. High pressure fuel union nut. 1219 1177 High pressure fuel union 1220 Fuel oil strainer grid. sleeve. 1221 Fuel oil strainer valve.

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Part No.	Part Name	Part No.	Part Name
1222	Fuel oil strainer stuffing box.	G1287	Day tank pump complete:
1223	Fuel oil strainer gauze.	14.3	Parts 1287, 1288, 1288A, 1289,
1224	Fuel oil strainer cover gasket		1291, 1292.
1225	Fuel oil strainer cover clamp.	1287	Day tank pump body.
1225A	Fuel oil strainer cover clamp	1288	Day tank pump plunger.
122011	capscrew.	. 1288A	Day tank pump plunger nut.
1005D	Fuel oil strainer cover clamp	1289	Day tank pump plunger eye.
1225B	stud.	1289A	Day tank pump plunger eye
1226	Fuel oil strainer grid spring.		pin.
G1230	Fuel relief valve complete.	1290	Day tank pump check valve.
1230	Fuel relief valve body.	1291	Day tank pump gland.
1231	Fuel relief valve gland.	1292	Day tank pump gland nut.
1232	Fuel relief valve stud.	1293	Day tank pump connecting
1233	Fuel relief valve stem valve.		rod.
1234	Fuel relief valve seat.	1295	Fuel day tank and flange.
1235	Fuel relief valve stud collar.	1297	Fuel day tank strainer cage
1236	Fuel relief valve spring.		and gauze.
1237	Fuel relief valve spring cage.	1306	Chain sprocket (driver on
1238	Fuel relief valve handle bear-	· Common	crank shaft).
1200	ing.	1307	Chain sprocket (driven on
1238A	Fuel relief valve handle bear-		pump shaft).
LEGULA	ing pin.	1308	Chain shield.
1239	Fuel relief valve handle ad-	1309	Chain shield cap.
1200	justing screw.	1310	Ball bearing container—blind
1240	Fuel relief valve handle plug	77	end.
2410	(lower).	1310A	Ball bearing container-shaft
1241	Fuel relief valve stud capnut.	A CONTRACTOR	end.
1242	Fuel relief valve fuel fitting.		Driven chain sprocket shaft.
G1243	Fuel relief valve handle com-	1312	Driven chain sprocket shaft
GILIO	plete.		ball bearing.
1243	Fuel relief valve handle cam.	1313	Silent chain.
1244	Fuel relief valve handle sector	1314	Silent chain adjusting screw.
1245	Fuel relief valve spring plug,	1315	Idler spindle.
	(top).	1315A	Idler spindle lock nut.
1247	Fuel relief valve stud collar		Idler spindle split ring.
1 1 1 1 1 1 1	gasket.	1316	Idler spindle bearing housing.
1248	Fuel relief valve stud gasket.	1317	Idler spindle chain sprocket.
1249	Governor speed control han-	1318	Idler spindle roller bearing
	dle screw.	2010	complete. Idler spindle steel cover.
G1275	Priming pump complete: Parts	1319	
EL RES	1275, 1276, 1276A, 1278 and	1319A	Idler spindle roller bearing
W	packing.	1320	Idler spillate toller assess
1275	Priming pump cylinder.	4004	spacer. Idler spindle spacer.
1276	Priming pump plunger.	1321	Idler spindle spacer washer.
1276/	Priming pump plunger nut,	1322	Idler spindle felt washer.
.1278	Priming pump cap.	1323	Idler spindle left wasnes
1279	Priming pump plunger eye.	1357	Lubricating oil service tank
1280	Priming pump plunger con-	11000	with flanges.
1, 252	necting rod.	1358	Lubricating oil service tank
1280	A Priming pump plunger pin.	Pi	collar. Also for fuel day tank
1281	Priming pump plunger handle	1359	Lubricating oil service tank
1282			cover. Also for fuel day talla
100 Land	bracket.	1360	Lubricating oil service tank
1282	A Priming pump plunger handle		cover plug. Also for fuel da
4000	bracket spacer.		tank.
1282	B Priming pump plunger handle	1362	Lubricating oil service tank
4000	bracket bar.		strainer cage and gauze.
1282	C Priming pump plunger handle	N. S. P. S. D.	
	bracket lever pin.		
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When ordering parts be sure to furnish Engine Number, as well as Part Number

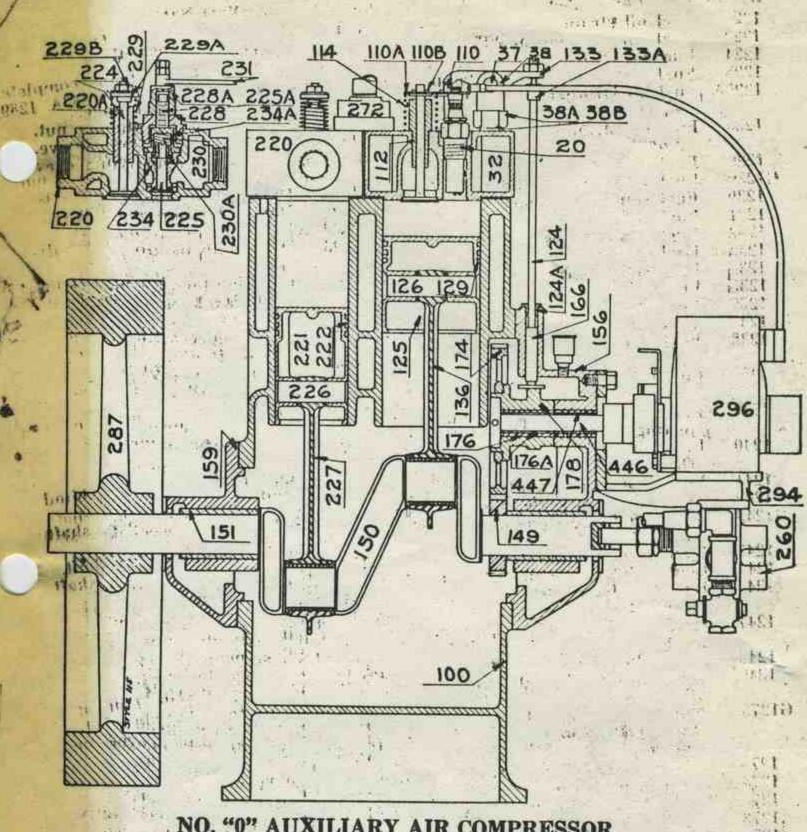
AUXILIARY COMPRESSOR

In ordering all repair parts for compressor BE SURE and give BOTH the part name and number, also ENGINE NUMBER of auxiliary.

	METATA	idi j.		
	20	Spark plug	220	Air pump head
	32	Engine cylinder head	220A	Air pump head valve-stem
	33	Engine and pump cylinder	ZZOII.	bushing bushing
		head stud	221	Air pump piston
	33A		222	
	UUIA	head stud, long	224	Air pump piston ring
	33B		225	Air pump inlet valve
	0013	Engine and pump cylinder head stud nut		Air pump outlet valve
	37		225A	Air pump outlet valve ad-
	38	Rocker pin	000	justing screw
		CALLED AND CONTROL BOURTS CONTROL CONT	226	Air pump piston pin
	384		227	Air pump connecting rod
	38B		228	Air pump valve nut
	100	Cylinder and frame	228A	Air pump outlet valve cap
	110	Inlet or exhaust valve	229	Air pump inlet valve spring
	110A	Inlet or exhaust valve Valve stem cup	229A	The state of the s
	110B	Valve stem cup lock		guide
	112	Valve stem bushing	229B	Air pump inlet valve spring
	114	Inlet valve spring Exhaust valve spring	The second	guide nut
	114A	Exhaust valve spring	230	Air pump outlet valve
	124	Exhaust rocker push rod		spring
	124A	Exhaust rocker push rod	230A	Air pump outlet valve
		bushing		spring guide
1	125	Engine piston	231	Air pump relief handle
	126	Engine piston pin	234	Air pump outlet valve
		Engine piston ring		bushing
	133	Exhaust rocker with ad-	234A	Air pump outlet valve
		justing screw	to the	bushing cap
	133A	Exhaust rocker ball socket	240	Carburetor
	136	Engine connecting rod	260	Rotary pump
	149	Crankshaft pinton	260A	Rotary pump cap screw
	150	Crank shaft	272	Water outlet flange
	151	Crankshaft bushing	287	Flywheel
	156	Crankshaft bearing	294	Magneto bracket
	159	Crankshaft bearing, fly-	296	Magneto
	100	wheel end	446	Magneto drive shaft
	160	Crankshaft bearing cap	447	Magneto drive shaft bush-
	200	screw bearing cap		ing
	163	Frame cover door	G2441	Lubricator assembled
		Frame cover door clamp	2441	Lubricator bottom with air
	163R	Frame cover door clamp	2441	
	TOOD	wing nut	2441A	pipe Lubricator valve
	163C	The state of the s		I Zeli Planik Digital Rigis (1985) / C
	1000	Frame cover door clamp	2441B	Lubricator spring
	166	Cam lifter	2441C	Lubricator washer and nut
	174		2442	Lubricator top with stem
11.		Cam gear	2442A	Lubricator clamp stud
	176	Cam gear pin	2442B	Lubricator clamp stud nut
	176A	Cam gear pin bushing	2443	Lubricator plug wik tem
	178	Exhaust cam and gear hub	2443A	Lubricator glass
	202	Breather pipe		
			100	

Always Give Part Name, Part Number and ENGINE NUMBER

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Always Give Part Name, Part Number and ENGINE NUMBER

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